

# Nachrichtlich

# MIKE 21 & MIKE 3 Flow Model FM

Hydrodynamic and Transport Module

Scientific Documentation



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## 1 Introduction

This document presents the scientific background for the new MIKE 21 & MIKE 3 Flow Model FM<sup>1</sup> modelling system developed by DHI Water & Environment. The objective is to provide the user with a detailed description of the flow and transport model equations, numerical discretization and solution methods. Also model validation is discussed in this document.

The MIKE 21 & MIKE 3 Flow Model FM is based on a flexible mesh approach and it has been developed for applications within oceanographic, coastal and estuarine environments. The modelling system may also be applied for studies of overland flooding.

The system is based on the numerical solution of the two/three-dimensional incompressible Reynolds averaged Navier-Stokes equations invoking the assumptions of Boussinesq and of hydrostatic pressure. Thus, the model consists of continuity, momentum, temperature, salinity and density equations and it is closed by a turbulent closure scheme. For the 3D model the free surface is taken into account using a sigma coordinate transformation approach.

The spatial discretization of the primitive equations is performed using a cell-centred finite volume method. The spatial domain is discretized by subdivision of the continuum into non-overlapping elements/cells. In the horizontal plane an unstructured grid is used while in the vertical domain in the 3D model a structured mesh is used. In the 2D model the elements can be triangles or quadrilateral elements. In the 3D model the elements can be prisms or bricks whose horizontal faces are triangles and quadrilateral elements, respectively.

<sup>&</sup>lt;sup>1</sup> Including the MIKE 21 Flow Model FM (two-dimensional flow) and MIKE 3 Flow Model FM (threedimensional flow)





## 2 Governing Equations

## 2.1 3D Governing Equations in Cartesian Coordinates

#### 2.1.1 Shallow water equations

The model is based on the solution of the three-dimensional incompressible Reynolds averaged Navier-Stokes equations, subject to the assumptions of Boussinesq and of hydrostatic pressure.

The local continuity equation is written as

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} + \frac{\partial w}{\partial z} = S$$
(2.1)

and the two horizontal momentum equations for the x- and y-component, respectively

$$\frac{\partial u}{\partial t} + \frac{\partial u^2}{\partial x} + \frac{\partial vu}{\partial y} + \frac{\partial wu}{\partial z} = fv - g \frac{\partial \eta}{\partial x} - \frac{1}{\rho_0} \frac{\partial p_a}{\partial x} - \frac{g}{\rho_0} \int_z^{\eta} \frac{\partial \rho}{\partial x} dz - \frac{1}{\rho_0 h} \left( \frac{\partial s_{xx}}{\partial x} + \frac{\partial s_{xy}}{\partial y} \right) + F_u + \frac{\partial}{\partial z} \left( v_t \frac{\partial u}{\partial z} \right) + u_s S$$
(2.2)

$$\frac{\partial v}{\partial t} + \frac{\partial v^2}{\partial y} + \frac{\partial uv}{\partial x} + \frac{\partial wv}{\partial z} = -fu - g\frac{\partial \eta}{\partial y} - \frac{1}{\rho_0}\frac{\partial p_a}{\partial y} - \frac{1}{\rho_0}\frac{\partial p_a}{\partial y} - \frac{g}{\rho_0}\int_z^{\eta}\frac{\partial \rho}{\partial y}dz - \frac{1}{\rho_0h}\left(\frac{\partial s_{yx}}{\partial x} + \frac{\partial s_{yy}}{\partial y}\right) + F_v + \frac{\partial}{\partial z}\left(v_t\frac{\partial v}{\partial z}\right) + v_sS$$
(2.3)

where *t* is the time; *x*, *y* and *z* are the Cartesian coordinates;  $\eta$  is the surface elevation; *d* is the still water depth;  $h = \eta + d$  is the total water depth; *u*, *v* and *w* are the velocity components in the *x*, *y* and *z* direction;  $f = 2\Omega \sin \phi$  is the Coriolis parameter ( $\Omega$  is the angular rate of revolution and  $\phi$  the geographic latitude); *g* is the gravitational acceleration;  $\rho$  is the density of water;  $s_{xx}, s_{xy}, s_{yx}$  and  $s_{yy}$  are components of the radiation stress tensor;  $v_t$  is the vertical turbulent (or eddy) viscosity;  $p_a$  is the atmospheric pressure;  $\rho_o$  is the reference density of water. *S* is the magnitude of the discharge due to point sources and  $(u_s, v_s)$  is the velocity by which the water is discharged into the ambient water. The horizontal stress terms are described using a gradient-stress relation, which is simplified to

$$F_{u} = \frac{\partial}{\partial x} \left( 2A \frac{\partial u}{\partial x} \right) + \frac{\partial}{\partial y} \left( A \left( \frac{\partial u}{\partial y} + \frac{\partial v}{\partial x} \right) \right)$$
(2.4)

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$$F_{v} = \frac{\partial}{\partial x} \left( A \left( \frac{\partial u}{\partial y} + \frac{\partial v}{\partial x} \right) \right) + \frac{\partial}{\partial y} \left( 2A \frac{\partial v}{\partial y} \right)$$
(2.5)

where A is the horizontal eddy viscosity.

The surface and bottom boundary condition for u, v and w are

At 
$$z = \eta$$
:  

$$\frac{\partial \eta}{\partial t} + u \frac{\partial \eta}{\partial x} + v \frac{\partial \eta}{\partial y} - w = 0, \quad \left(\frac{\partial u}{\partial z}, \frac{\partial v}{\partial z}\right) = \frac{1}{\rho_0 v_t} (\tau_{sx}, \tau_{sy})$$
(2.6)  
At  $z = -d$ :  

$$u \frac{\partial d}{\partial x} + v \frac{\partial d}{\partial y} + w = 0, \quad \left(\frac{\partial u}{\partial z}, \frac{\partial v}{\partial z}\right) = \frac{1}{\rho_0 v_t} (\tau_{bx}, \tau_{by})$$
(2.7)

where  $(\tau_{sx}, \tau_{sy})$  and  $(\tau_{bx}, \tau_{by})$  are the *x* and *y* components of the surface wind and bottom stresses.

The total water depth, h, can be obtained from the kinematic boundary condition at the surface, once the velocity field is known from the momentum and continuity equations. However, a more robust equation is obtained by vertical integration of the local continuity equation

$$\frac{\partial h}{\partial t} + \frac{\partial h\overline{u}}{\partial x} + \frac{\partial h\overline{v}}{\partial y} = hS + \widehat{P} - \widehat{E}$$
(2.8)

where  $\hat{P}$  and  $\hat{E}$  are precipitation and evaporation rates, respectively, and  $\overline{u}$  and  $\overline{v}$  are the depth-averaged velocities

$$h\overline{u} = \int_{-d}^{\eta} u dz , \quad h\overline{v} = \int_{-d}^{\eta} v dz$$
(2.9)

The fluid is assumed to be incompressible. Hence, the density,  $\rho$ , does not depend on the pressure, but only on the temperature, T, and the salinity, s, via the equation of state

$$\rho = \rho(T, s) \tag{2.10}$$

Here the UNESCO equation of state is used (see UNESCO, 1981).



#### 2.1.2 Transport equations for salt and temperature

The transports of temperature, T, and salinity, s, follow the general transport-diffusion equations as

$$\frac{\partial T}{\partial t} + \frac{\partial uT}{\partial x} + \frac{\partial vT}{\partial y} + \frac{\partial wT}{\partial z} = F_T + \frac{\partial}{\partial z} \left( D_v \frac{\partial T}{\partial z} \right) + \hat{H} + T_s S$$
(2.11)

$$\frac{\partial s}{\partial t} + \frac{\partial us}{\partial x} + \frac{\partial vs}{\partial y} + \frac{\partial ws}{\partial z} = F_s + \frac{\partial}{\partial z} \left( D_v \frac{\partial s}{\partial z} \right) + s_s S$$
(2.12)

where  $D_v$  is the vertical turbulent (eddy) diffusion coefficient.  $\hat{H}$  is a source term due to heat exchange with the atmosphere.  $T_s$  and  $s_s$  are the temperature and the salinity of the source. *F* are the horizontal diffusion terms defined by

$$(F_T, F_s) = \left[\frac{\partial}{\partial x} \left(D_h \frac{\partial}{\partial x}\right) + \frac{\partial}{\partial y} \left(D_h \frac{\partial}{\partial y}\right)\right] (T, s)$$
(2.13)

where  $D_h$  is the horizontal diffusion coefficient. The diffusion coefficients can be related to the eddy viscosity

$$D_h = \frac{A}{\sigma_T}$$
 and  $D_v = \frac{v_t}{\sigma_T}$  (2.14)

where  $\sigma_T$  is the Prandtl number. In many applications a constant Prandtl number can be used (see Rodi (1984)).

The surface and bottom boundary conditions for the temperature are

At 
$$z = \eta$$
:  

$$D_{h} \frac{\partial T}{\partial z} = \frac{Q_{n}}{\rho_{0}c_{p}} + T_{p}\hat{P} - T_{e}\hat{E}$$
(2.15)  
At  $z = -d$ :  

$$\frac{\partial T}{\partial z} = 0$$
(2.16)

where  $Q_n$  is the surface net heat flux and  $c_p = 4217 J/(kg \cdot {}^{\circ}K)$  is the specific heat of the water. A detailed description for determination of  $\hat{H}$  and  $Q_n$  is given in Section 2.10.



The surface and bottom boundary conditions for the salinity are

At 
$$z = \eta$$
:  
 $\frac{\partial s}{\partial z} = 0$   
At  $z = -d$ :  
(2.17)

$$\frac{\partial s}{\partial z} = 0 \tag{2.18}$$

When heat exchange from the atmosphere is included, the evaporation is defined as

$$\widehat{E} = \begin{cases} \frac{q_{\nu}}{\rho_0 l_{\nu}} & q_{\nu} > 0\\ 0 & q_{\nu} \le 0 \end{cases}$$
(2.19)

where  $q_v$  is the latent heat flux and  $l_v = 2.5 \cdot 10^6$  is the latent heat of vaporisation of water.

#### 2.1.3 Transport equation for a scalar quantity

The conservation equation for a scalar quantity is given by

$$\frac{\partial C}{\partial t} + \frac{\partial uC}{\partial x} + \frac{\partial vC}{\partial y} + \frac{\partial wC}{\partial z} = F_C + \frac{\partial}{\partial z} \left( D_v \frac{\partial C}{\partial z} \right) - k_p C + C_s S$$
(2.20)

where *C* is the concentration of the scalar quantity,  $k_p$  is the linear decay rate of the scalar quantity,  $C_s$  is the concentration of the scalar quantity at the source and  $D_v$  is the vertical diffusion coefficient.  $F_c$  is the horizontal diffusion term defined by

$$F_{C} = \left[\frac{\partial}{\partial x} \left(D_{h} \frac{\partial}{\partial x}\right) + \frac{\partial}{\partial y} \left(D_{h} \frac{\partial}{\partial y}\right)\right] C$$
(2.21)

where  $D_h$  is the horizontal diffusion coefficient.

#### 2.1.4 Turbulence model

The turbulence is modelled using an eddy viscosity concept. The eddy viscosity is often described separately for the vertical and the horizontal transport. Here several turbulence models can be applied: a constant viscosity, a vertically parabolic viscosity and a standard k- $\varepsilon$  model (Rodi, 1984). In many numerical simulations the small-scale turbulence cannot be resolved with the chosen spatial resolution. This kind of turbulence can be approximated using sub-grid scale models.



#### Vertical eddy viscosity

The eddy viscosity derived from the log-law is calculated by

$$\nu_t = U_\tau h \left( c_1 \frac{z+d}{h} + c_2 \left( \frac{z+d}{h} \right)^2 \right)$$
(2.22)

where  $U_{\tau} = \max(U_{\tau s}, U_{\tau b})$  and  $c_1$  and  $c_2$  are two constants.  $U_{\tau s}$  and  $U_{\tau b}$  are the friction velocities associated with the surface and bottom stresses,  $c_1 = 0.41$  and  $c_2 = -0.41$  give the standard parabolic profile.

In applications with stratification the effects of buoyancy can be included explicitly. This is done through the introduction of a Richardson number dependent damping of the eddy viscosity coefficient, when a stable stratification occurs. The damping is a generalisation of the Munk-Anderson formulation (Munk and Anderson, 1948)

$$v_t = v_t^* (1 + aRi)^{-b}$$
(2.23)

where  $v_t^*$  is the undamped eddy viscosity and *Ri* is the local gradient Richardson number

$$Ri = -\frac{g}{\rho_0} \frac{\partial \rho}{\partial z} \left( \left( \frac{\partial u}{\partial z} \right)^2 + \left( \frac{\partial v}{\partial z} \right)^2 \right)^{-1}$$
(2.24)

a = 10 and b = 0.5 are empirical constants.

In the k- $\varepsilon$  model the eddy-viscosity is derived from turbulence parameters k and  $\varepsilon$  as

$$v_t = c_\mu \frac{k^2}{\varepsilon} \tag{2.25}$$

where *k* is the turbulent kinetic energy per unit mass (TKE),  $\varepsilon$  is the dissipation of TKE and  $c_{\mu}$  is an empirical constant.

The turbulent kinetic energy, k, and the dissipation of TKE,  $\varepsilon$ , are obtained from the following transport equations

$$\frac{\partial k}{\partial t} + \frac{\partial uk}{\partial x} + \frac{\partial vk}{\partial y} + \frac{\partial wk}{\partial z} = F_k + \frac{\partial}{\partial z} \left( \frac{\nu_t}{\sigma_k} \frac{\partial k}{\partial z} \right) + P + B - \varepsilon$$
(2.26)

$$\frac{\partial \varepsilon}{\partial t} + \frac{\partial u\varepsilon}{\partial x} + \frac{\partial v\varepsilon}{\partial y} + \frac{\partial w\varepsilon}{\partial z} = F_{\varepsilon} + \frac{\partial}{\partial z} \left( \frac{v_t}{\sigma_{\varepsilon}} \frac{\partial \varepsilon}{\partial z} \right) + \frac{\varepsilon}{k} (c_{1\varepsilon}P + c_{3\varepsilon}B - c_{2\varepsilon}\varepsilon)$$
(2.27)

where the shear production, P, and the buoyancy production, B, are given as



$$P = \frac{\tau_{xz}}{\rho_0} \frac{\partial u}{\partial z} + \frac{\tau_{yz}}{\rho_0} \frac{\partial v}{\partial z} \approx v_t \left( \left( \frac{\partial u}{\partial z} \right)^2 + \left( \frac{\partial v}{\partial z} \right)^2 \right)$$
(2.28)

$$B = -\frac{v_t}{\sigma_t} N^2 \tag{2.29}$$

with the Brunt-Väisälä frequency, N, defined by

$$N^2 = -\frac{g}{\rho_0} \frac{\partial \rho}{\partial z}$$
(2.30)

 $\sigma_t$  is the turbulent Prandtl number and  $\sigma_k$ ,  $\sigma_{\varepsilon}$ ,  $c_{1\varepsilon}$ ,  $c_{2\varepsilon}$  and  $c_{3\varepsilon}$  are empirical constants. *F* are the horizontal diffusion terms defined by

$$(F_k, F_{\varepsilon}) = \left[\frac{\partial}{\partial x} \left(D_h \frac{\partial}{\partial x}\right) + \frac{\partial}{\partial y} \left(D_h \frac{\partial}{\partial y}\right)\right](k, \varepsilon)$$
(2.31)

The horizontal diffusion coefficients are given by  $D_h = A/\sigma_k$  and  $D_h = A/\sigma_{\varepsilon}$ , respectively.

Several carefully calibrated empirical coefficients enter the k-e turbulence model. The empirical constants are listed in (2.47) (see Rodi, 1984).

#### Table 2.1 Empirical constants in the k- $\varepsilon$ model.

C <sub>µ</sub>	$c_{1\varepsilon}$	C <sub>2ε</sub>	$c_{3\varepsilon}$	$\sigma_t$	$\sigma_k$	$\sigma_{\varepsilon}$
0.09	1.44	1.92	0	0.9	1.0	1.3

At the surface the boundary conditions for the turbulent kinetic energy and its rate of dissipation depend on the wind shear,  $U_{rs}$ 

At 
$$z = \eta$$
:

$$k = \frac{1}{\sqrt{c_{\mu}}} U_{\tau s}^2$$

$$\varepsilon = \frac{U_{\tau s}^3}{\kappa \Delta z_b} \qquad \qquad \text{for} \quad U_{\tau s} > 0$$

$$\frac{\partial k}{\partial z} = 0 \qquad \varepsilon = \frac{\left(k\sqrt{c_{\mu}}\right)^{3/2}}{a\kappa h} \qquad \text{for} \quad U_{\pi} = 0 \tag{2.33}$$

(2.32)



where  $\kappa = 0.4$  is the von Kármán constant, a = 0.07 is and empirical constant and  $\Delta z_s$  is the distance from the surface where the boundary condition is imposed. At the seabed the boundary conditions are

At 
$$z = -d$$
:  

$$k = \frac{1}{\sqrt{c_{\mu}}} U_{\tau b}^{2} \qquad \qquad \varepsilon = \frac{U_{\tau b}^{3}}{\kappa \Delta z_{b}} \qquad (2.34)$$

where  $\Delta z_b$  is the distance from the bottom where the boundary condition is imposed.

#### Horizontal eddy viscosity

In many applications a constant eddy viscosity can be used for the horizontal eddy viscosity. Alternatively, Smagorinsky (1963) proposed to express sub-grid scale transports by an effective eddy viscosity related to a characteristic length scale. The subgrid scale eddy viscosity is given by

$$A = c_s^2 l^2 \sqrt{2S_{ij} S_{ij}}$$
(2.35)

where  $c_s$  is a constant, *I* is a characteristic length and the deformation rate is given by

$$S_{ij} = \frac{1}{2} \left( \frac{\partial u_i}{\partial x_j} + \frac{\partial u_j}{\partial x_i} \right) \quad (i, j = 1, 2)$$
(2.36)

#### 2.1.5 Governing equations in Cartesian and sigma coordinates

The equations are solved using a vertical  $\sigma$ -transformation

$$\sigma = \frac{z - z_b}{h}, \quad x' = x, \quad y' = y$$
 (2.37)

where  $\sigma$  varies between 0 at the bottom and 1 at the surface. The coordinate transformation implies relations such as

$$\frac{\partial}{\partial z} = \frac{1}{h} \frac{\partial}{\partial \sigma}$$
(2.38)

$$\left(\frac{\partial}{\partial x}, \frac{\partial}{\partial y}\right) = \left(\frac{\partial}{\partial x'} - \frac{1}{h} \left(-\frac{\partial d}{\partial x} + \sigma \frac{\partial h}{\partial x}\right) \frac{\partial}{\partial \sigma}, \frac{\partial}{\partial y'} - \frac{1}{h} \left(-\frac{\partial d}{\partial y} + \sigma \frac{\partial h}{\partial y}\right) \frac{\partial}{\partial \sigma}\right)$$
(2.39)

In this new coordinate system the governing equations are given as

$$\frac{\partial h}{\partial t} + \frac{\partial hu}{\partial x'} + \frac{\partial hv}{\partial y'} + \frac{\partial h\omega}{\partial \sigma} = hS$$
(2.40)



$$\frac{\partial hu}{\partial t} + \frac{\partial hu^2}{\partial x'} + \frac{\partial hvu}{\partial y'} + \frac{\partial h\omega u}{\partial \sigma} = fvh - gh\frac{\partial \eta}{\partial x'} - \frac{h}{\rho_0}\frac{\partial p_a}{\partial x'} - \frac{hg}{\rho_0}\int_z^{\eta}\frac{\partial \rho}{\partial x}dz - \frac{1}{\rho_0}\left(\frac{\partial s_{xx}}{\partial x} + \frac{\partial s_{xy}}{\partial y}\right) + hF_u + \frac{\partial}{\partial\sigma}\left(\frac{v_v}{h}\frac{\partial u}{\partial\sigma}\right) + hu_sS$$
(2.41)

$$\frac{\partial hv}{\partial t} + \frac{\partial huv}{\partial x'} + \frac{\partial hv^2}{\partial y'} + \frac{\partial hav}{\partial \sigma} = -fuh - gh\frac{\partial \eta}{\partial y'} - \frac{h}{\rho_0}\frac{\partial p_a}{\partial y'} - \frac{hg}{\rho_0}\int_z^{\eta}\frac{\partial \rho}{\partial y}dz - \frac{1}{\rho_0}\left(\frac{\partial s_{yx}}{\partial x} + \frac{\partial s_{yy}}{\partial y}\right) + hF_v + \frac{\partial}{\partial\sigma}\left(\frac{v_v}{h}\frac{\partial v}{\partial\sigma}\right) + hv_sS$$
(2.42)

$$\frac{\partial hT}{\partial t} + \frac{\partial huT}{\partial x'} + \frac{\partial hvT}{\partial y'} + \frac{\partial h\omega T}{\partial \sigma} = hF_T + \frac{\partial}{\partial \sigma} \left(\frac{D_v}{h} \frac{\partial T}{\partial \sigma}\right) + h\hat{H} + hT_s S$$
(2.43)

$$\frac{\partial hs}{\partial t} + \frac{\partial hus}{\partial x'} + \frac{\partial hvs}{\partial y'} + \frac{\partial h\omega s}{\partial \sigma} = hF_s + \frac{\partial}{\partial \sigma} \left( \frac{D_v}{h} \frac{\partial s}{\partial \sigma} \right) + hs_s S$$
(2.44)

$$\frac{\partial hk}{\partial t} + \frac{\partial huk}{\partial x'} + \frac{\partial hvk}{\partial y'} + \frac{\partial h\omega k}{\partial \sigma} = hF_k + \frac{1}{h}\frac{\partial}{\partial \sigma} \left(\frac{\nu_t}{\sigma_k}\frac{\partial k}{\partial \sigma}\right) + h(P + B - \varepsilon)$$
(2.45)

$$\frac{\partial h\varepsilon}{\partial t} + \frac{\partial hu\varepsilon}{\partial x'} + \frac{\partial hv\varepsilon}{\partial y'} + \frac{\partial h\omega\varepsilon}{\partial \sigma} = hF_{\varepsilon} + \frac{1}{h}\frac{\partial}{\partial\sigma}\left(\frac{\nu_{\iota}}{\sigma_{\varepsilon}}\frac{\partial\varepsilon}{\partial\sigma}\right) + h\frac{\varepsilon}{k}\left(c_{1\varepsilon}P + c_{3\varepsilon}B - c_{2\varepsilon}\varepsilon\right)$$
(2.46)

$$\frac{\partial hC}{\partial t} + \frac{\partial huC}{\partial x'} + \frac{\partial hvC}{\partial y'} + \frac{\partial h\omega C}{\partial \sigma} = hF_C + \frac{\partial}{\partial \sigma} \left(\frac{D_v}{h}\frac{\partial C}{\partial \sigma}\right) - hk_pC + hC_sS$$
(2.47)

The modified vertical velocity is defined by

$$\omega = \frac{1}{h} \left[ w + u \frac{\partial d}{\partial x'} + v \frac{\partial d}{\partial y'} - \sigma \left( \frac{\partial h}{\partial t} + u \frac{\partial h}{\partial x'} + v \frac{\partial h}{\partial y'} \right) \right]$$
(2.48)

The modified vertical velocity is the velocity across a level of constant  $\sigma.$  The horizontal diffusion terms are defined as

$$hF_{u} \approx \frac{\partial}{\partial x} \left( 2hA \frac{\partial u}{\partial x} \right) + \frac{\partial}{\partial y} \left( hA \left( \frac{\partial u}{\partial y} + \frac{\partial v}{\partial x} \right) \right)$$
(2.49)



$$hF_{\nu} \approx \frac{\partial}{\partial x} \left( hA \left( \frac{\partial u}{\partial y} + \frac{\partial v}{\partial x} \right) \right) + \frac{\partial}{\partial y} \left( 2hA \frac{\partial v}{\partial y} \right)$$
(2.50)

$$h(F_T, F_s, F_k, F_{\varepsilon}, F_c) \approx \left[\frac{\partial}{\partial x} \left(hD_h \frac{\partial}{\partial x}\right) + \frac{\partial}{\partial y} \left(hD_h \frac{\partial}{\partial y}\right)\right] (T, s, k, \varepsilon, C)$$
(2.51)

The boundary condition at the free surface and at the bottom are given as follows

At  $\sigma$  =1:

$$\omega = 0, \quad \left(\frac{\partial u}{\partial \sigma}, \frac{\partial v}{\partial \sigma}\right) = \frac{h}{\rho_0 v_t} \left(\tau_{sx}, \tau_{sy}\right) \tag{2.52}$$

At  $\sigma$  =0:

$$\omega = 0, \quad \left(\frac{\partial u}{\partial \sigma}, \frac{\partial v}{\partial \sigma}\right) = \frac{h}{\rho_0 v_t} \left(\tau_{bx}, \tau_{by}\right) \tag{2.53}$$

The equation for determination of the water depth is not changed by the coordinate transformation. Hence, it is identical to Eq. (2.6).

## 2.2 3D Governing Equations in Spherical and Sigma Coordinates

In spherical coordinates the independent variables are the longitude,  $\lambda$ , and the latitude,  $\phi$ . The horizontal velocity field (*u*,*v*) is defined by

$$u = R\cos\phi\frac{d\lambda}{dt} \qquad \qquad v = R\frac{d\phi}{dt} \tag{2.54}$$

where *R* is the radius of the earth.

In this coordinate system the governing equations are given as (all superscripts indicating the horizontal coordinate in the new coordinate system are dropped in the following for notational convenience)

$$\frac{\partial h}{\partial t} + \frac{1}{R\cos\phi} \left( \frac{\partial hu}{\partial \lambda} + \frac{\partial hv\cos\phi}{\partial \phi} \right) + \frac{\partial h\omega}{\partial \sigma} = hS$$
(2.55)



$$\frac{\partial hu}{\partial t} + \frac{1}{R\cos\phi} \left( \frac{\partial hu^2}{\partial \lambda} + \frac{\partial hvu\cos\phi}{\partial \phi} \right) + \frac{\partial h\omega u}{\partial \sigma} = \left( f + \frac{u}{R}\tan\phi \right) vh - \frac{1}{R\cos\phi} \left( gh \frac{\partial \eta}{\partial \lambda} + \frac{1}{\rho_0} \frac{\partial p_a}{\partial \lambda} + \frac{g}{\rho_0} \int_{z}^{\eta} \frac{\partial \rho}{\partial \lambda} dz + \frac{1}{\rho_0} \left( \frac{\partial s_{xx}}{\partial \lambda} + \cos\phi \frac{\partial s_{xy}}{\partial \phi} \right) \right) + hF_u + \frac{\partial}{\partial \sigma} \left( \frac{v_v}{h} \frac{\partial u}{\partial \sigma} \right) + hu_s S$$

$$(2.56)$$

$$\frac{\partial hv}{\partial t} + \frac{1}{R\cos\phi} \left( \frac{\partial huv}{\partial\lambda} + \frac{\partial hv^{2}\cos\phi}{\partial\phi} \right) + \frac{\partial h\omega v}{\partial\sigma} = -\left( f + \frac{u}{R}\tan\phi \right) uh - \frac{1}{R} \left( gh \frac{\partial \eta}{\partial\phi} + \frac{1}{\rho_{0}} \frac{\partial p_{a}}{\partial\phi} + \frac{g}{\rho_{0}} \int_{z}^{\eta} \frac{\partial \rho}{\partial\phi} dz + \frac{1}{\rho_{0}} \left( \frac{1}{\cos\phi} \frac{\partial s_{yx}}{\partial\lambda} + \frac{\partial s_{yy}}{\partial\phi} \right) \right) + hF_{v} + \frac{\partial}{\partial\sigma} \left( \frac{v_{v}}{h} \frac{\partial v}{\partial\sigma} \right) + hv_{s}S$$

$$(2.57)$$

$$\frac{\partial hT}{\partial t} + \frac{1}{R\cos\phi} \left( \frac{\partial huT}{\partial \lambda} + \frac{\partial hvT\cos\phi}{\partial \phi} \right) + \frac{\partial h\omega T}{\partial \sigma} = hF_T + \frac{\partial}{\partial \sigma} \left( \frac{D_v}{h} \frac{\partial T}{\partial \sigma} \right) + h\hat{H} + hT_s S$$
(2.58)

$$\frac{\partial hs}{\partial t} + \frac{1}{R\cos\phi} \left( \frac{\partial hus}{\partial \lambda} + \frac{\partial hvs\cos\phi}{\partial \phi} \right) + \frac{\partial h\omega s}{\partial \sigma} = hF_s + \frac{\partial}{\partial \sigma} \left( \frac{D_v}{h} \frac{\partial s}{\partial \sigma} \right) + hs_s S$$
(2.59)

$$\frac{\partial hk}{\partial t} + \frac{1}{R\cos\phi} \left( \frac{\partial huk}{\partial\lambda} + \frac{\partial hvk\cos\phi}{\partial\phi} \right) + \frac{\partial h\omega k}{\partial\sigma} = hF_k + \frac{1}{h} \frac{\partial}{\partial\sigma} \left( \frac{v_t}{\sigma_k} \frac{\partial k}{\partial\sigma} \right) + h(P + B - \varepsilon)$$
(2.60)

$$\frac{\partial h\varepsilon}{\partial t} + \frac{1}{R\cos\phi} \left( \frac{\partial hu\varepsilon}{\partial\lambda} + \frac{\partial hv\varepsilon\cos\phi}{\partial\phi} \right) + \frac{\partial h\omega\varepsilon}{\partial\sigma} = hF_{\varepsilon} + \frac{1}{h} \frac{\partial}{\partial\sigma} \left( \frac{\nu_{\iota}}{\sigma_{\varepsilon}} \frac{\partial\varepsilon}{\partial\sigma} \right) + h\frac{\varepsilon}{k} \left( c_{1\varepsilon}P + c_{3\varepsilon}B - c_{2\varepsilon}\varepsilon \right)$$
(2.61)

$$\frac{\partial hC}{\partial t} + \frac{1}{R\cos\phi} \left( \frac{\partial huC}{\partial \lambda} + \frac{\partial hvC\cos\phi}{\partial \phi} \right) + \frac{\partial h\omega C}{\partial \sigma} = hF_{c} + \frac{\partial}{\partial \sigma} \left( \frac{D_{v}}{h} \frac{\partial C}{\partial \sigma} \right) - hk_{p}C + hC_{s}S$$
(2.62)

The modified vertical velocity in spherical coordinates is defined by



$$\omega = \frac{1}{h} \left[ w + \frac{u}{R\cos\phi} \frac{\partial d}{\partial\lambda} + \frac{v}{R} \frac{\partial d}{\partial y} - \sigma \left( \frac{\partial h}{\partial t} + \frac{u}{R\cos\phi} \frac{\partial h}{\partial\lambda} + \frac{v}{R} \frac{\partial h}{\partial\phi} \right) \right]$$
(2.63)

The equation determining the water depth in spherical coordinates is given as

$$\frac{\partial h}{\partial t} + \frac{1}{R\cos\phi} \left( \frac{\partial h\overline{u}}{\partial\lambda} + \frac{\partial h\overline{v}\cos\phi}{\partial\phi} \right) = hS$$
(2.64)

## 2.3 2D Governing Equations in Cartesian Coordinates

#### 2.3.1 Shallow water equations

Integration of the horizontal momentum equations and the continuity equation over depth  $h = \eta + d$  the following two-dimensional shallow water equations are obtained

$$\frac{\partial h}{\partial t} + \frac{\partial h\overline{u}}{\partial x} + \frac{\partial h\overline{v}}{\partial y} = hS$$
(2.65)

$$\frac{\partial h\overline{u}}{\partial t} + \frac{\partial h\overline{u}^{2}}{\partial x} + \frac{\partial h\overline{vu}}{\partial y} = f\overline{v}h - gh\frac{\partial\eta}{\partial x} - \frac{h}{\rho_{0}}\frac{\partial\rho}{\partial x} - \frac{h}{\rho_{0}}\frac{\partial\rho}{\partial x} - \frac{gh^{2}}{\partial x}\frac{\partial\rho}{\partial x} + \frac{\tau_{sx}}{\rho_{0}} - \frac{\tau_{bx}}{\rho_{0}} - \frac{1}{\rho_{0}}\left(\frac{\partial s_{xx}}{\partial x} + \frac{\partial s_{xy}}{\partial y}\right) + \frac{\partial}{\partial x}(hT_{xx}) + \frac{\partial}{\partial y}(hT_{xy}) + hu_{s}S$$
(2.66)

$$\frac{\partial h\overline{v}}{\partial t} + \frac{\partial h\overline{u}\overline{v}}{\partial x} + \frac{\partial h\overline{v}^{2}}{\partial y} = -f\overline{u}h - gh\frac{\partial \eta}{\partial y} - \frac{h}{\rho_{0}}\frac{\partial p_{a}}{\partial y} - \frac{gh^{2}}{\rho_{0}}\frac{\partial \rho}{\partial y} + \frac{\tau_{sy}}{\rho_{0}} - \frac{\tau_{by}}{\rho_{0}} - \frac{1}{\rho_{0}}\left(\frac{\partial s_{yx}}{\partial x} + \frac{\partial s_{yy}}{\partial y}\right) + \frac{\partial}{\partial x}\left(hT_{xy}\right) + \frac{\partial}{\partial y}\left(hT_{yy}\right) + hv_{s}S$$
(2.67)

The overbar indicates a depth average value. For example,  $\overline{u}$  and  $\overline{v}$  are the depthaveraged velocities defined by

$$h\overline{u} = \int_{-d}^{\eta} u dz , \quad h\overline{v} = \int_{-d}^{\eta} v dz$$
(2.68)

The lateral stresses  $T_{ij}$  include viscous friction, turbulent friction and differential advection. They are estimated using an eddy viscosity formulation based on of the depth average velocity gradients



$$T_{xx} = 2A \frac{\partial \overline{u}}{\partial x}, \quad T_{xy} = A \left( \frac{\partial \overline{u}}{\partial y} + \frac{\partial \overline{v}}{\partial x} \right), \quad T_{yy} = 2A \frac{\partial \overline{v}}{\partial y}$$
 (2.69)

## 2.3.2 Transport equations for salt and temperature

Integrating the transport equations for salt and temperature over depth the following twodimensional transport equations are obtained

$$\frac{\partial h\overline{T}}{\partial t} + \frac{\partial h\overline{u}\overline{T}}{\partial x} + \frac{\partial h\overline{v}\overline{T}}{\partial y} = hF_T + h\widehat{H} + hT_sS$$
(2.70)

$$\frac{\partial h\overline{s}}{\partial t} + \frac{\partial h\overline{u}\overline{s}}{\partial x} + \frac{\partial h\overline{v}\overline{s}}{\partial y} = hF_s + hs_s S$$
(2.71)

where  $\overline{T}$  and  $\overline{s}$  is the depth average temperature and salinity.

#### 2.3.3 Transport equations for a scalar quantity

Integrating the transport equations for a scalar quantity over depth the following twodimensional transport equations are obtained

$$\frac{\partial h\overline{C}}{\partial t} + \frac{\partial h\overline{u}\overline{C}}{\partial x} + \frac{\partial h\overline{v}\overline{C}}{\partial y} = hF_C - hk_p\overline{C} + hC_sS$$
(2.72)

where  $\overline{C}$  is the depth average scalar quantity.

## 2.4 2D Governing Equations in Spherical Coordinates

In spherical coordinates the independent variables are the longitude,  $\lambda$  ,and the latitude,  $\phi$ . The horizontal velocity field (*u*,*v*) is defined by

$$\overline{u} = R\cos\phi\frac{d\lambda}{dt}$$
  $\overline{v} = R\frac{d\phi}{dt}$  (2.73)

where *R* is the radius of the earth.

In spherical coordinates the governing equation can be written

$$\frac{\partial h}{\partial t} + \frac{1}{R\cos\phi} \left( \frac{\partial h\overline{u}}{\partial\lambda} + \frac{\partial h\overline{v}\cos\phi}{\partial\phi} \right) = 0$$
(2.74)



$$\frac{\partial h\overline{u}}{\partial t} + \frac{1}{R\cos\phi} \left( \frac{\partial h\overline{u}^2}{\partial \lambda} + \frac{\partial h\overline{v}\overline{u}\cos\phi}{\partial \phi} \right) = \left( f + \frac{\overline{u}}{R}\tan\phi \right) \overline{v}h$$
$$- \frac{1}{R\cos\phi} \left( gh \frac{\partial \eta}{\partial \lambda} - \frac{h}{\rho_0} \frac{\partial p_a}{\partial \lambda} + \frac{gh^2}{2\rho_0} \frac{\partial \rho}{\partial \lambda} + \frac{1}{\rho_0} \left( \frac{\partial s_{xx}}{\partial \lambda} + \cos\phi \frac{\partial s_{xy}}{\partial \phi} \right) \right) +$$
$$\frac{\tau_{xx}}{\rho_0} - \frac{\tau_{bx}}{\rho_0} + \frac{\partial}{\partial x} \left( hT_{xx} \right) + \frac{\partial}{\partial y} \left( hT_{xy} \right) + hu_s S$$
(2.75)

$$\frac{\partial h\overline{v}}{\partial t} + \frac{1}{R\cos\phi} \left( \frac{\partial h\overline{u}\overline{v}}{\partial\lambda} + \frac{\partial h\overline{v}^{2}\cos\phi}{\partial t} \right) = -\left( f + \frac{\overline{u}}{R}\tan\phi \right) \overline{u}h$$

$$-\frac{1}{R} \left( gh \frac{\partial \eta}{\partial \phi} - \frac{h}{\rho_{0}} \frac{\partial p_{a}}{\partial \phi} + \frac{gh^{2}}{2\rho_{0}} \frac{\partial \rho}{\partial \phi} + \frac{1}{\rho_{0}} \left( \frac{1}{\cos\phi} \frac{\partial s_{yx}}{\partial\lambda} + \frac{\partial s_{yy}}{\partial \phi} \right) \right) +$$

$$\frac{\tau_{sy}}{\rho_{0}} - \frac{\tau_{by}}{\rho_{0}} + \frac{\partial}{\partial x} \left( hT_{xy} \right) + \frac{\partial}{\partial y} \left( hT_{yy} \right) + hv_{s}S$$
(2.76)

$$\frac{\partial h\overline{T}}{\partial t} + \frac{1}{R\cos\phi} \left( \frac{\partial h\overline{u}\overline{T}}{\partial\lambda} + \frac{\partial h\overline{v}\overline{T}\cos\phi}{\partial\phi} \right) = hF_T + h\widehat{H} + hT_s S$$
(2.77)

$$\frac{\partial h\overline{s}}{\partial t} + \frac{1}{R\cos\phi} \left( \frac{\partial h\overline{u}\overline{s}}{\partial\lambda} + \frac{\partial h\overline{v}\overline{s}\cos\phi}{\partial\phi} \right) = hF_s + hs_s S$$
(2.78)

$$\frac{\partial h\overline{C}}{\partial t} + \frac{1}{R\cos\phi} \left( \frac{\partial h\overline{u}\overline{C}}{\partial\lambda} + \frac{\partial h\overline{v}\overline{C}\cos\phi}{\partial\phi} \right) = hF_C - hk_p\overline{C} + hC_sS$$
(2.79)

## 2.5 Bottom Stress

The bottom stress,  $\vec{\tau}_b = (\tau_{bx}, \tau_{by})$ , is determined by a quadratic friction law

$$\frac{\vec{\tau}_b}{\rho_0} = c_f \vec{u}_b |\vec{u}_b| \tag{2.80}$$

where  $c_f$  is the drag coefficient and  $\vec{u}_b = (u_b, v_b)$  is the flow velocity above the bottom. The friction velocity associated with the bottom stress is given by

$$U_{\tau b} = \sqrt{c_f \left| u_b \right|^2} \tag{2.81}$$

For two-dimensional calculations  $\vec{u}_b$  is the depth-average velocity and the drag coefficient can be determined from the Chezy number, C, or the Manning number, M

$$c_f = \frac{g}{C^2} \tag{2.82}$$

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$$c_f = \frac{g}{\left(Mh^{1/6}\right)^2}$$
(2.83)

For three-dimensional calculations  $\vec{u}_b$  is the velocity at a distance  $\Delta z_b$  above the sea bed and the drag coefficient is determined by assuming a logarithmic profile between the seabed and a point  $\Delta z_b$  above the seabed

$$c_f = \frac{1}{\left(\frac{1}{\kappa} \ln\left(\frac{\Delta z_b}{z_0}\right)\right)^2}$$
(2.84)

where  $\kappa = 0.4$  is the von Kármán constant and  $z_0$  is the bed roughness length scale. When the boundary surface is rough,  $z_0$ , depends on the roughness height,  $k_s$ 

$$z_0 = mk_s \tag{2.85}$$

where m is approximately 1/30.

Note, that the Manning number can be estimated from the bed roughness length using the following

$$M = \frac{25.4}{k_s^{1/6}} \tag{2.86}$$

The wave induced bed resistance can be determined from

$$c_f = \left(\frac{u_{fc}}{u_b}\right)^2 \tag{2.87}$$

where  $U_{fc}$  is the friction velocity calculated by considering the conditions in the wave boundary layer. For a detailed description of the wave induced bed resistance, see Fredsøe (1984) and Jones et.al. (2014).

#### 2.6 Wind Stress

In areas not covered by ice the surface stress,  $\vec{\tau}_s = (\tau_{sx}, \tau_{sy})$ , is determined by the winds above the surface. The stress is given by the following empirical relation

$$\overline{\tau}_{s} = \rho_{a} c_{d} |u_{w}| \overline{u}_{w}$$
(2.88)

where  $\rho_a$  is the density of air,  $c_d$  is the drag coefficient of air, and  $\vec{u}_w = (u_w, v_w)$  is the wind speed 10 m above the sea surface. The friction velocity associated with the surface stress is given by



$$U_{\tau s} = \sqrt{\frac{\rho_a c_f \left| \overline{u}_w \right|^2}{\rho_0}} \tag{2.89}$$

The drag coefficient can either be a constant value or depend on the wind speed. The empirical formula proposed by Wu (1980, 1994) is used for the parameterisation of the drag coefficient.

$$c_{f} = \begin{cases} c_{a} & w_{10} < w_{a} \\ c_{a} + \frac{c_{b} - c_{a}}{w_{b} - w_{a}} (w_{10} - w_{a}) & w_{a} \le w_{10} < w_{b} \\ c_{b} & w_{10} \ge w_{b} \end{cases}$$
(2.90)

where  $c_a$ ,  $c_b$ ,  $w_a$  and  $w_b$  are empirical factors and  $w_{10}$  is the wind velocity 10 m above the sea surface. The default values for the empirical factors are  $c_a = 1.255 \cdot 10^{-3}$ ,  $c_b = 2.425 \cdot 10^{-3}$ ,  $w_a = 7$  m/s and  $w_b = 25$  m/s. These give generally good results for open sea applications. Field measurements of the drag coefficient collected over lakes indicate that the drag coefficient is larger than open ocean data. For a detailed description of the drag coefficient see Geernaert and Plant (1990).

## 2.7 Ice Coverage

It is possible to take into account the effects of ice coverage on the flow field.

In areas where the sea is covered by ice the wind stress is excluded. Instead, the surface stress is caused by the ice roughness. The surface stress,  $\vec{\tau}_s = (\tau_{sx}, \tau_{sy})$ , is determined by a quadratic friction law

$$\frac{\vec{r}_s}{\rho_0} = c_f \vec{u}_s \left| \vec{u}_s \right| \tag{2.91}$$

where  $c_f$  is the drag coefficient and  $\vec{u}_s = (u_s, v_s)$  is the flow velocity below the surface. The friction velocity associated with the surface stress is given by

$$U_{\tau s} = \sqrt{c_f \left| u_s \right|^2} \tag{2.92}$$

For two-dimensional calculations  $\vec{u}_s$  is the depth-average velocity and the drag coefficient can be determined from the Manning number, M

$$c_f = \frac{g}{\left(Mh^{1/6}\right)^2}$$
(2.93)

The Manning number is estimated from the bed roughness length using the following





For three-dimensional calculations  $\vec{u}_s$  is the velocity at a distance  $\Delta z_s$  below the surface and the drag coefficient is determined by assuming a logarithmic profile between the surface and a point  $\Delta z_b$  below the surface

$$c_f = \frac{1}{\left(\frac{1}{\kappa} \ln\left(\frac{\Delta z_s}{z_0}\right)\right)^2}$$
(2.95)

where  $\kappa = 0.4$  is the von Kármán constant and  $z_0$  is the bed roughness length scale. When the boundary surface is rough,  $z_0$ , depends on the roughness height,  $k_s$ 

$$z_0 = mk_s \tag{2.96}$$

where m is approximately 1/30.

#### 2.8 Tidal Potential

The tidal potential is a force, generated by the variations in gravity due to the relative motion of the earth, the moon and the sun that act throughout the computational domain. The forcing is expanded in frequency space and the potential considered as the sum of a number of terms each representing different tidal constituents. The forcing is implemented as a so-called equilibrium tide, which can be seen as the elevation that theoretically would occur, provided the earth was covered with water. The forcing enters the momentum equations (e.g. (2.66) or (2.75)) as an additional term representing the gradient of the equilibrium tidal elevations, such that the elevation  $\eta$  can be seen as the sum of the actual elevation and the equilibrium tidal potential.

$$\eta = \eta_{ACTUAL} + \eta_T \tag{2.97}$$

The equilibrium tidal potential  $\eta_T$  is given as

$$\eta_T = \sum_i e_i H_i f_i L_i \cos(2\pi \frac{t}{T_i} + b_i + i_0 x)$$
(2.98)

where  $\eta_T$  is the equilibrium tidal potential, *i* refers to constituent number (note that the constituents here are numbered sequentially),  $e_i$  is a correction for earth tides based on Love numbers,  $H_i$  is the amplitude,  $f_i$  is a nodal factor,  $L_i$  is given below, *t* is time,  $T_i$  is the period of the constituent,  $b_i$  is the phase and *x* is the longitude of the actual position.

The phase *b* is based on the motion of the moon and the sun relative to the earth and can be given by

$$b_{i} = (i_{1} - i_{0})s + (i_{2} + i_{0})h + i_{3}p + i_{4}N + i_{5}p_{s} + u_{i}\sin(N)$$
(2.99)



where  $i_0$  is the species,  $i_1$  to  $i_5$  are Doodson numbers, u is a nodal modulation factor (see Table 2.3) and the astronomical arguments s, h, p, N and  $p_s$  are given in Table 2.2.

#### Table 2.2Astronomical arguments (Pugh, 1987)

Mean longitude of the moon	s	277.02+481267.89T+0.0011T <sup>2</sup>
Mean longitude of the sun	h	280.19+36000.77T+0.0003T <sup>2</sup>
Longitude of lunar perigee	р	334.39+4069.04T-0.0103T <sup>2</sup>
Longitude of lunar ascending node	N	259.16-1934.14T+0.0021T <sup>2</sup>
Longitude of perihelion	ps	281.22+1.72T+0.0005T <sup>2</sup>

In Table 2.2 the time, T, is in Julian century from January 1 1900 UTC, thus T = (365(y - 1900) + (d - 1) + i)/36525 and i = int (y-1901)/4), y is year and d is day number

L depends on species number  $i_0$  and latitude y as

$i_0 = 0$	$L = 3\sin^2(y) - 1$
$i_0 = 1$	$L = \sin(2y)$
$i_0 = 2$	$L = \cos^2(y)$

The nodal factor  $f_i$  represents modulations to the harmonic analysis and can for some constituents be given as shown in Table 2.3.

	f <sub>i</sub>	Ui
M <sub>m</sub>	1.000 - 0.130 cos(N)	0
M <sub>f</sub>	1.043 + 0.414 cos(N)	-23.7 sin(N)
Q <sub>1,</sub> O <sub>1</sub>	1.009 + 0.187 cos(N)	10.8 sin(N)
K <sub>1</sub>	1.006 + 0.115 cos(N)	-8.9 sin(N)
$2N_2, \mu_2, \nu_2, N_2, M_2$	1.000 - 0.037 cos(N)	-2.1 sin(N)
K <sub>2</sub>	1.024 + 0.286 cos(N)	-17.7 sin(N)

#### Table 2.3Nodal modulation terms (Pugh, 1987)

#### 2.9 Wave Radiation

The second order stresses due to breaking of short period waves can be included in the simulation. The radiation stresses act as driving forces for the mean flow and can be used to calculate wave induced flow. For 3D simulations a simple approach is used. Here a uniform variation is used for the vertical variation in radiation stress.



#### 2.10 Heat Exchange

The heat exchange with the atmosphere is calculated on basis of the four physical processes

- Latent heat flux (or the heat loss due to vaporisation)
- Sensible heat flux (or the heat flux due to convection)
- Net short wave radiation
- Net long wave radiation

Latent and sensible heat fluxes and long-wave radiation are assumed to occur at the surface. The absorption profile for the short-wave flux is approximated using Beer's law. The attenuation of the light intensity is described through the modified Beer's law as

$$I(d) = (1 - \beta)I_0 e^{-\lambda d}$$
(2.100)

where I(d) is the intensity at depth *d* below the surface;  $I_0$  is the intensity just below the water surface;  $\beta$  is a quantity that takes into account that a fraction of light energy (the infrared) is absorbed near the surface;  $\lambda$  is the light extinction coefficient. Typical values for  $\beta$  and  $\lambda$  are 0.2-0.6 and 0.5-1.4 m<sup>-1</sup>, respectively.  $\beta$  and  $\lambda$  are user-specified constants. The default values are  $\beta = 0.3$  and  $\lambda = 1.0 m^{-1}$ . The fraction of the light energy that is absorbed near the surface is  $\beta I_0$ . The net short-wave radiation,  $q_{sr,net}$ , is attenuated as described by the modified Beer's law. Hence the surface net heat flux is given by

$$Q_{n} = q_{v} + q_{c} + \beta q_{sr,net} + q_{lr,net}$$
(2.101)

For three-dimensional calculations the source term  $\hat{H}$  is given by

$$\widehat{H} = \frac{\partial}{\partial z} \left( \frac{q_{sr,net} (1 - \beta) e^{-\lambda(\eta - z)}}{\rho_0 c_p} \right) = \frac{q_{sr,net} (1 - \beta) \frac{e^{-\lambda(\eta - z)}}{\lambda}}{\rho_0 c_p}$$
(2.102)

For two-dimensional calculations the source term  $\hat{H}$  is given by

$$\widehat{H} = \frac{q_v + q_c + q_{sr,net} + q_{lr,net}}{\rho_0 c_p}$$
(2.103)

The calculation of the latent heat flux, sensible heat flux, net short wave radiation, and net long wave radiation as described in the following sections.

In areas covered by ice the heat exchange is excluded.

#### 2.10.1 Vaporisation

Dalton's law yields the following relationship for the vaporative heat loss (or latent flux), see Sahlberg, 1984



$$q_{v} = LC_{e}(a_{1} + b_{1}W_{2m})(Q_{water} - Q_{air})$$
(2.104)

where  $L = 2.5 \cdot 10^6 J/kg$  is the latent heat vaporisation (in the literature  $L = 2.5 \cdot 10^6 - 2300 T_{water}$  is commonly used);  $C_e = 1.32 \cdot 10^{-3}$  is the moisture transfer coefficient (or Dalton number);  $W_{2m}$  is the wind speed 2 m above the sea surface;  $Q_{water}$  is the water vapour density close to the surface;  $Q_{air}$  is the water vapour density in the atmosphere;  $a_1$  and  $b_1$  are user specified constants. The default values are  $a_1 = 0.5$  and  $b_1 = 0.9$ .

Measurements of  $Q_{water}$  and  $Q_{air}$  are not directly available but the vapour density can be related to the vapour pressure as

$$Q_i = \frac{0.2167}{T_i + T_k} e_i \tag{2.105}$$

in which subscript *i* refers to both water and air. The vapour pressure close to the sea,  $e_{water}$ , can be expressed in terms of the water temperature assuming that the air close to the surface is saturated and has the same temperature as the water

$$e_{water} = 6.11e^{K} \left( \frac{1}{T_k} - \frac{1}{T_{water} + T_k} \right)$$
(2.106)

where K = 5418 °K and  $T_K = 273.15 \text{ °}K$  is the temperature at 0 C. Similarly the vapour pressure of the air,  $e_{air}$ , can be expressed in terms of the air temperature and the relative humidity, R

$$e_{air} = R \cdot 6.11 e^{K} \left( \frac{1}{T_k} - \frac{1}{T_{air} + T_k} \right)$$
(2.107)

Replacing  $Q_{water}$  and  $Q_{air}$  with these expressions the latent heat can be written as

$$q_{v} = -P_{v}\left(a_{1} + b_{1}W_{2m}\right) \cdot \left(\frac{\exp\left(K\left(\frac{1}{T_{k}} - \frac{1}{T_{water} + T_{k}}\right)\right)}{T_{water} + T_{k}} - \frac{R \cdot \exp\left(K\left(\frac{1}{T_{k}} - \frac{1}{T_{air} + T_{k}}\right)\right)}{T_{air} + T_{k}}\right)$$

$$(2.108)$$

where all constants have been included in a new latent constant  $P_v = 4370 J \cdot {}^{\circ}K / m^3$ . During cooling of the surface the latent heat loss has a major effect with typical values up to 100 W/m<sup>2</sup>.



The wind speed,  $W_2$ , 2 m above the sea surface is calculated from the from the wind speed,  $W_{10}$ , 10 m above the sea surface using the following formula:

Assuming a logarithmic profile the wind speed, u(z), at a distance z above the sea surface is given by

$$u(z) = \frac{u_*}{\kappa} \log\left(\frac{z}{z_o}\right)$$
(2.109)

where  $u_*$  is the wind friction velocity,  $z_0$  is the sea roughness and  $\kappa = 0.4$  is von Karman's constant.  $u_*$  and  $z_0$  are given by

$$z_0 = z_{Charnock} \mu_*^2 / g \tag{2.110}$$

$$u_* = \frac{\kappa u(z)}{\log\left(\frac{z}{z_0}\right)}$$
(2.111)

where  $z_{Charnock}$  is the Charnock parameter. The default value is  $z_{Charnock} = 0.014$ . The wind speed,  $W_2$ , 2 m above the sea surface is then calculated from the from the wind speed,  $W_{10}$ , 10m above the sea surface by first solving Eq. (2.114) and Eq. (2.115) iteratively for  $z_0$  with z=10m and  $u(z)=W_{10}$ . Then  $W_2$  is given by

( )

$$W_{2} = W_{10} \frac{\log\left(\frac{2}{z_{o}}\right)}{\log\left(\frac{10}{z_{0}}\right)} \qquad \qquad W_{10} > 0.5m/s$$

$$W_{2} = W_{10} \qquad \qquad W_{10} \le 0.5m/s$$
(2.112)

The heat loss due to vaporization occurs both by wind driven forced convection by and free convection. The effect of free convection is taken into account by the parameter  $a_1$  in Eq. (2.104). The free convection is also taken into account by introducing a critical wind speed  $W_{critical}$  so that the wind speed used in Eq. (2.112) is obtained as  $W_{10}=max(W_{10},W_{critical})$ . The default value for the critical wind speed is 2 m/s.

#### 2.10.2 Convection

The sensible heat flux,  $q_c (W/m^2)$ , (or the heat flux due to convection) depends on the type of boundary layer between the sea surface and the atmosphere. Generally this boundary layer is turbulent implying the following relationship

$$q_{c} = \begin{cases} \rho_{air} c_{air} c_{heating} W_{10} (T_{air} - T_{water}) & T_{air} \ge T \\ \rho_{air} c_{air} c_{cooling} W_{10} (T_{air} - T_{water}) & T_{air} < T \end{cases}$$

$$(2.113)$$



where  $\rho_{air}$  is the air density 1.225 kg/m<sup>3</sup>;  $c_{air} = 1007 J/(kg \cdot {}^{\circ}K)$  is the specific heat of air;  $c_{heating} = 0.0011$  and  $c_{cooling} = 0.0011$ , respectively, is the sensible transfer coefficient (or Stanton number) for heating and cooling (see Kantha and Clayson, 2000);  $W_{10}$  is the wind speed 10 m above the sea surface;  $T_{water}$  is the temperature at the sea surface;  $T_{air}$  is the temperature of the air.

The convective heat flux typically varies between 0 and 100 W/m<sup>2</sup>.

The heat loss due to convection occurs both by wind driven forced convection by and free convection. The free convection is taken into account by introducing a critical wind speed  $W_{critical}$  so that the wind speed used in Eq. (2.113) is obtained as  $W_{10}=max(W_{10},W_{critical})$ . The default value for the critical wind speed is 2 m/s.

#### 2.10.3 Short wave radiation

Radiation from the sun consists of electromagnetic waves with wave lengths varying from 1,000 to 30,000 Å. Most of this is absorbed in the ozone layer, leaving only a fraction of the energy to reach the surface of the Earth. Furthermore, the spectrum changes when sunrays pass through the atmosphere. Most of the infrared and ultraviolet compound is absorbed such that the solar radiation on the Earth mainly consists of light with wave lengths between 4,000 and 9,000 Å. This radiation is normally termed short wave radiation. The intensity depends on the distance to the sun, declination angle and latitude, extraterrestrial radiation and the cloudiness and amount of water vapour in the atmosphere (see lqbal, 1983)

The eccentricity in the solar orbit,  $E_0$ , is given by

$$E_0 = \left(\frac{r_0}{r}\right)^2 = 1.000110 + 0.034221\cos(\Gamma) + 0.001280\sin(\Gamma) + 0.000719\cos(2\Gamma) + 0.000077\sin(2\Gamma)$$
(2.114)

where  $r_0$  is the mean distance to the sun, *r* is the actual distance and the day angle  $\Gamma$  (*rad*) is defined by

$$\Gamma = \frac{2\pi (d_n - 1)}{365} \tag{2.115}$$

and  $d_n$  is the Julian day of the year.

The daily rotation of the Earth around the polar axes contributes to changes in the solar radiation. The seasonal radiation is governed by the declination angle,  $\delta$  (*rad*), which can be expressed by

$$\delta = 0.006918 - 0.399912\cos(\Gamma) + 0.07257\sin(\Gamma) - 0.006758\cos(2\Gamma) + 0.000907\sin(2\Gamma) - (2.116)$$
  
0.002697\cos(3\Gamma) + 0.00148sin(3\Gamma)



The day length,  $n_d$ , varies with  $\delta$ . For a given latitude,  $\phi$ , (positive on the northern hemisphere) the day length is given by

$$n_d = \frac{24}{\pi} \arccos\left(-\tan(\phi)\tan(\delta)\right) \tag{2.117}$$

and the sunrise angle,  $\omega_{sr}~(rad)$  , and the sunset angle  $\omega_{ss}~(rad)$  are

$$\omega_{sr} = \arccos\left(-\tan(\phi)\tan(\delta)\right) \quad and \quad \omega_{ss} = -\omega_{sr}$$
(2.118)

The intensity of short wave radiation on the surface parallel to the surface of the Earth changes with the angle of incidence. The highest intensity is in zenith and the lowest during sunrise and sunset. Integrated over one day the extraterrestrial intensity,

 $H_0 (MJ/m^2/day)$ , in short wave radiation on the surface can be derived as

$$H_0 = \frac{24}{\pi} q_{sc} E_0 \cos(\phi) \cos(\delta) (\sin(\omega_{sr}) - \omega_{sr} \cos(\omega_{sr}))$$
(2.119)

where  $q_{sc} = 4.9212 (MJ/m^2/h)$  is the solar constant.

For determination of daily radiation under cloudy skies,  $H (MJ/m^2/day)$ , the following relation is used

$$\frac{H}{H_0} = a_2 + b_2 \frac{n}{n_d}$$
(2.120)

in which n is the number of sunshine hours and  $n_d$  is the maximum number of sunshine hours.  $a_2$  and  $b_2$  are user specified constants. The default values are  $a_2 = 0.295$  and  $b_2 = 0.371$ . The user-specified clearness coefficient corresponds to  $n/n_d$  . Thus the solar radiation,  $q_s (W/m^2)$ , can be expressed as

$$q_{s} = \left(\frac{H}{H_{0}}\right) q_{0} \left(a_{3} + b_{3} \cos\left(\omega_{i}\right)\right) \frac{10^{6}}{3600}$$
(2.121)

where

$$a_3 = 0.4090 + 0.5016 \sin\left(\omega_{sr} - \frac{\pi}{3}\right)$$
(2.122)

$$b_3 = 0.6609 + 0.4767 \sin\left(\omega_{sr} - \frac{\pi}{3}\right)$$
(2.123)

The extraterrestrial intensity,  $q_0 (MJ/m^2/h)$  and the hour angle  $\omega_i$  is given by



$$q_0 = q_{sc} E_0 \left( \sin(\phi) \sin(\delta) + \frac{24}{\pi} \cos(\phi) \cos(\delta) \cos(\omega_i) \right)$$
(2.124)

$$\omega_{i} = \frac{\pi}{12} \left( 12 + \Delta t_{displacement} + \frac{4}{60} (L_{S} - L_{E}) - \frac{E_{t}}{60} - t_{local} \right)$$
(2.125)

 $\Delta t_{displacement}$  is the displacement hours due to summer time and the time meridian  $L_s$  is the standard longitude for the time zone.  $\Delta t_{displacement}$  and  $L_s$  are user specified constants. The default values are  $\Delta t_{displacement} = 0$  (h) and  $L_s = 0$  (deg).  $L_E$  is the local longitude in degrees.  $E_t$  (s) is the discrepancy in time due to solar orbit and is varying during the year. It is given by

$$E_{\tau} = \begin{pmatrix} 0.000075 + 0.001868\cos(\Gamma) - 0.032077\sin(\Gamma) \\ -0.014615\cos(2\Gamma) - 0.04089\sin(2\Gamma) \end{pmatrix} \cdot 229.18$$
(2.126)

Finally,  $t_{local}$  is the local time in hours.

Solar radiation that impinges on the sea surface does not all penetrate the water surface. Parts are reflected back and are lost unless they are backscattered from the surrounding atmosphere. This reflection of solar energy is termed the albedo. The amount of energy, which is lost due to albedo, depends on the angle of incidence and angle of refraction. For a smooth sea the reflection can be expressed as

$$\alpha = \frac{1}{2} \left( \frac{\sin^2(i-r)}{\sin^2(i+r)} + \frac{\tan^2(i-r)}{\tan^2(i+r)} \right)$$
(2.127)

where *i* is the angle of incidence, *r* the refraction angle and  $\alpha$  the reflection coefficient, which typically varies from 5 to 40 %.  $\alpha$  can be approximated using

$$\alpha = \begin{cases} \frac{altitude}{5} \ 0.48 & altitude < 5\\ \frac{30 - altitude}{25} (0.48 - 0.05) & 5 \le altitude \le 30\\ 0.05 & altitude > 30 \end{cases}$$
(2.128)

where the altitude in degrees is given by

$$altitude = 90 - \left(\frac{180}{\pi}\arccos(\sin(\delta)\sin(\phi) + \cos(\delta)\cos(\phi)\cos(\omega_i))\right)$$
(2.129)

Thus the net short wave radiation,  $q_{s,net}$  ( $W/m^2$ ), can possibly be expressed as

$$q_{sr,net} = (1 - \alpha)q_s \tag{2.130}$$



The net short wave radiation,  $q_{sr,net}$ , can be calculated using empirical formulae as described above. Alternatively, the net short wave radiation can be calculated using Eq. (2.130) where the solar radiation,  $q_s$ , is specified by the user or the net short wave radiation,  $q_{sr,net}$ , can be given by the user.

#### 2.10.4 Long wave radiation

A body or a surface emits electromagnetic energy at all wavelengths of the spectrum. The long wave radiation consists of waves with wavelengths between 9,000 and 25,000 Å. The radiation in this interval is termed infrared radiation and is emitted from the atmosphere and the sea surface. The long wave emittance from the surface to the atmosphere minus the long wave radiation from the atmosphere to the sea surface is called the net long wave radiation and is dependent on the cloudiness, the air temperature, the vapour pressure in the air and the relative humidity. The net outgoing long wave radiation,  $q_{lr,net}$  ( $W/m^2$ ), is given by Brunt's equation (See Lind and Falkenmark, 1972)

$$q_{lr,net} = -\sigma_{sb} \left( T_{air} + T_K \right)^4 \left( a - b \sqrt{e_d} \left( c + d \frac{n}{n_d} \right) \right)$$
(2.131)

where  $e_d$  is the vapour pressure at dew point temperature measured in *mb*; *n* is the number of sunshine hours,  $n_d$  is the maximum number of sunshine hours;

 $\sigma_{sb} = 5.6697 \cdot 10^{-8} W / (m^2 \cdot {}^{\circ}K^4)$  is Stefan Boltzman's constant;  $T_{air}$  (°*C*) is the air temperature. The coefficients *a*, *b*, *c* and *d* are given as

$$a = 0.56; b = 0.077 m b^{-\frac{1}{2}}; c = 0.10; d = .90$$
(2.132)

The vapour pressure is determined as

$$e_d = 10 \cdot R \ e_{saturated} \tag{2.133}$$

where *R* is the relative humidity and the saturated vapour pressure,  $e_{saturated}$  (*kPa*), with 100 % relative humidity in the interval from -51 to 52 °C can be estimated by

$$e_{saturated} = 3.38639 \cdot \left( \left( 7.38 \cdot 10^{-3} \cdot T_{air} + 0.8072 \right)^8 - 1.9 \cdot 10^{-5} \left| 1.8 \cdot T_{air} + 48 \right| + 1.316 \cdot 10^{-3} \right)$$
(2.134)

The net long wave radiation,  $q_{lr,net}$ , can be calculated using empirical formulae as described above. Alternatively, the net long wave radiation can be calculated as

$$q_{lr,net} = q_{ar,net} - q_{br}$$
(2.135)

where the net incident atmospheric radiation,  $q_{ar,net}$ , is specified by the user and the back radiation,  $q_{br}$ , is given by

$$q_{br} = (1 - r) \mathcal{E} \sigma_{sb} T_K^4 \tag{2.136}$$



where r=0.03 is the reflection coefficient and  $\epsilon$ =0.985 is the emissivity factor of the atmosphere. The net long wave radiation can also be specified by the user.



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## 3 Numerical Solution

## 3.1 Spatial Discretization

The discretization in solution domain is performed using a finite volume method. The spatial domain is discretized by subdivision of the continuum into non-overlapping cells/elements.

In the two-dimensional case the elements can be arbitrarily shaped polygons, however, here only triangles and quadrilateral elements are considered.

In the three-dimensional case a layered mesh is used: in the horizontal domain an unstructured mesh is used while in the vertical domain a structured mesh is used (see Figure 3.1). The vertical mesh is based on either sigma coordinates or combined sigma/z-level coordinates. For the hybrid sigma/z-level mesh sigma coordinates are used from the free surface to a specified depth and z-level coordinates are used below. The different types of vertical mesh are illustrated in Figure 3.2. The elements in the sigma domain and the z-level domain can be prisms with either a 3-sided or 4-sided polygonal base. Hence, the horizontal faces are either triangles or quadrilateral element. The elements are perfectly vertical and all layers have identical topology.



Figure 3.1 Principle of meshing for the three-dimensional case







The most important advantage using sigma coordinates is their ability to accurately represent the bathymetry and provide consistent resolution near the bed. However, sigma coordinates can suffer from significant errors in the horizontal pressure gradients, advection and mixing terms in areas with sharp topographic changes (steep slopes). These errors can give rise to unrealistic flows.

The use of z-level coordinates allows a simple calculation of the horizontal pressure gradients, advection and mixing terms, but the disadvantages are their inaccuracy in representing the bathymetry and that the stair-step representation of the bathymetry can result in unrealistic flow velocities near the bottom.



#### 3.1.1 Vertical Mesh

For the vertical discretization both a standard sigma mesh and a combined sigma/z-level mesh can be used. For the hybrid sigma/z-level mesh sigma coordinates are used from the free surface to a specified depth,  $z_{\sigma}$ , and z-level coordinates are used below. At least one sigma layer is needed to allow changes in the surface elevation.

#### Sigma

In the sigma domain a constant number of layers,  $N_{\sigma}$ , are used and each sigma layer is a fixed fraction of the total depth of the sigma layer,  $h_{\sigma}$ , where  $h_{\sigma} = \eta - \max(z_b, z_{\sigma})$ . The discretization in the sigma domain is given by a number of discrete  $\sigma$ -levels { $\sigma_i$ ,  $i = 1, (N_{\sigma} + 1)$ }. Here  $\sigma$  varies from  $\sigma_1 = 0$  at the bottom interface of the lowest sigma layer to  $\sigma_{N_{\sigma}+1} = 1$  at the free surface.

Variable sigma coordinates can be obtained using a discrete formulation of the general vertical coordinate (s-coordinate) system proposed by Song and Haidvogel (1994). First an equidistant discretization in a s-coordinate system ( $-1 \le s \le 0$ ) is defined

$$s_i = -\frac{N_{\sigma} + 1 - i}{N_{\sigma}}$$
  $i = 1, (N_{\sigma} + 1)$  (3.1)

The discrete sigma coordinates can then be determined by

$$\sigma_{i} = 1 + \sigma_{c} s_{i} + (1 - \sigma_{c}) c(s_{i}) \quad i = 1, (N_{\sigma} + 1)$$
(3.2)

where

$$c(s) = (1-b)\frac{\sinh(\theta s)}{\sinh(\theta)} + b\frac{\tanh\left(\theta\left(s+\frac{1}{2}\right)\right) - \tanh(\frac{\theta}{2})}{2\tanh(\frac{\theta}{2})}$$
(3.3)

Here  $\sigma_c$  is a weighting factor between the equidistant distribution and the stretch distribution,  $\theta$  is the surface control parameter and *b* is the bottom control parameter. The range for the weighting factor is  $0 < \sigma_c \le 1$  where the value 1 corresponds to equidistant distribution and 0 corresponds to stretched distribution. A small value of  $\sigma_c$  can result in linear instability. The range of the surface control parameter is  $0 < \theta \le 20$  and the range of the bottom control parameter is  $0 \le b \le 1$ . If  $\theta << 1$  and b=0 an equidistant vertical resolution is obtained. By increasing the value of the  $\theta$ , the highest resolution is achieved near the surface. If  $\theta > 0$  and b=1 a high resolution is obtained both near the surface and near the bottom.

Examples of a mesh using variable vertical discretization are shown in Figure 3.3 and Figure 3.4.





Figure 3.3 Example of vertical distribution using layer thickness distribution. Number of layers: 10, thickness of layers 1 to 10: .025, 0.075, 0.1, 0.01, 0.02, 0.02, 0.1, 0.1, 0.075, 0.025



Figure 3.4 Example of vertical distribution using variable distribution. Number of layers: 10,  $\sigma_c = 0.1$ ,  $\theta = 5$ , b = 1

#### Combined sigma/z-level

In the z-level domain the discretization is given by a number of discrete z-levels  $\{z_i, i = 1, (N_z + 1)\}$ , where  $N_z$  is the number of layers in the z-level domain.  $z_1$  is the minimum z-level and  $z_{N_z+1}$  is the maximum z-level, which is equal to the sigma depth,  $z_{\sigma}$ . The corresponding layer thickness is given by

$$\Delta z_i = z_{i+1} - z_i \qquad i = 1, N_z \tag{3.4}$$

The discretization is illustrated in Figure 3.5 and Figure 3.6.

Using standard z-level discretization the bottom depth is rounded to the nearest z-level. Hence, for a cell in the horizontal mesh with the cell-averaged depth,  $z_b$ , the cells in the corresponding column in the z-domain are included if the following criteria is satisfied



$$(z_{i+1} - z_i)/2 \ge z_b \quad i = 1, N_z$$
 (3.5)

The cell-averaged depth,  $z_b$ , is calculated as the mean value of the depth at the vortices of each cell. For the standard z-level discretization the minimum depth is given by  $z_l$ . Too take into account the correct depth for the case where the bottom depth is below the minimum z-level ( $z_1 > z_b$ ) a bottom fitted approach is used. Here, a correction factor,  $f_l$ , for the layer thickness in the bottom cell is introduced. The correction factor is used in the calculation of the volume and face integrals. The correction factor for the bottom cell is calculated by

$$f_1 = \frac{(z_2 - z_b)}{\Delta z_1} \tag{3.6}$$

The corrected layer thickness is given by  $\Delta z_1^* = f_1 \Delta z_1$ . The simple bathymetry adjustment approach is illustrated in Figure 3.5.

For a more accurate representation of the bottom depth an advanced bathymetry adjustment approach can be used. For a cell in the horizontal mesh with the cell-averaged depth,  $z_b$ , the cells in the corresponding column in the z-domain are included if the following criteria is satisfied

$$z_{i+1} > z_b$$
  $i = 1, N_z$  (3.7)

A correction factor,  $f_i$ , is introduced for the layer thickness

$$f_{i} = max \left( \frac{(z_{i+1} - z_{b})}{\Delta z_{i}}, \frac{z_{min}}{\Delta z_{i}} \right) \qquad z_{i} < z_{b} < z_{i+1} \text{ or } z_{1} > z_{b}$$

$$f_{i} = 1 \qquad z_{1} \ge z_{b}$$
(3.8)

A minimum layer thickness,  $\Delta z_{min}$ , is introduced to avoid very small values of the correction factor. The correction factor is used in the calculation of the volume and face integrals. The corrected layer thicknesses are given by { $\Delta z_i^* = f_i \Delta z_i$ ,  $i = 1, N_z$ }. The advanced bathymetry adjustment approach is illustrated in Figure 3.6.



Figure 3.5 Simple bathymetry adjustment approach





Figure 3.6 Advanced bathymetry adjustment approach

#### 3.1.2 Shallow water equations

The integral form of the system of shallow water equations can in general form be written

$$\frac{\partial U}{\partial t} + \nabla \cdot F(U) = S(U)$$
(3.9)

where **U** is the vector of conserved variables, **F** is the flux vector function and **S** is the vector of source terms.

In Cartesian coordinates the system of 2D shallow water equations can be written

$$\frac{\partial \boldsymbol{U}}{\partial t} + \frac{\partial \left(\boldsymbol{F}_{x}^{I} - \boldsymbol{F}_{x}^{V}\right)}{\partial x} + \frac{\partial \left(\boldsymbol{F}_{y}^{I} - \boldsymbol{F}_{y}^{V}\right)}{\partial y} = \boldsymbol{S}$$
(3.10)

where the superscripts *I* and *V* denote the inviscid (convective) and viscous fluxes, respectively and where



$$\begin{aligned} \boldsymbol{U} &= \begin{bmatrix} h\\ h\overline{u}\\ h\overline{v}\\ h\overline{v} \end{bmatrix}, \\ \boldsymbol{F}_{x}^{I} &= \begin{bmatrix} h\overline{u}\\ h\overline{u}^{2} + \frac{1}{2}g(h^{2} - d^{2})\\ h\overline{u}\overline{v} \end{bmatrix}, \quad \boldsymbol{F}_{x}^{I'} &= \begin{bmatrix} 0\\ hA\left(2\frac{\partial\overline{u}}{\partial x}\right)\\ hA\left(\frac{\partial\overline{u}}{\partial y} + \frac{\partial\overline{v}}{\partial x}\right) \end{bmatrix} \\ \boldsymbol{F}_{y}^{I} &= \begin{bmatrix} h\overline{v}\\ h\overline{v}u\\ h\overline{v}^{2} + \frac{1}{2}g(h^{2} - d^{2})\\ h\overline{v}^{2} + \frac{1}{2}g(h^{2} - d^{2}) \end{bmatrix}_{a}, \quad \boldsymbol{F}_{y}^{I'} &= \begin{bmatrix} 0\\ hA\left(\frac{\partial\overline{u}}{\partial y} + \frac{\partial\overline{v}}{\partial x}\right)\\ hA\left(2\frac{\partial\overline{v}}{\partial x}\right) \end{bmatrix} \end{aligned}$$
(3.11)  
$$\boldsymbol{S} &= \begin{bmatrix} 0\\ g\eta \frac{\partial d}{\partial x} + f\overline{v}h - \frac{h}{\rho_{0}}\frac{\partial p}{\partial x} - \frac{gh^{2}}{2\rho_{0}}\frac{\partial\rho}{\partial x} - \frac{1}{\rho_{0}}\left(\frac{\partial s_{xx}}{\partial x} + \frac{\partial s_{xy}}{\partial y}\right)\\ &+ \frac{\tau_{xx}}{\rho_{0}} - \frac{\tau_{hx}}{\rho_{0}} + hu_{s}\\ g\eta \frac{\partial d}{\partial y} - f\overline{u}h - \frac{h}{\rho_{0}}\frac{\partial p}{\partial y} - \frac{gh^{2}}{2\rho_{0}}\frac{\partial\rho}{\partial y} - \frac{1}{\rho_{0}}\left(\frac{\partial s_{yx}}{\partial x} + \frac{\partial s_{yy}}{\partial y}\right)\\ &+ \frac{\tau_{xy}}{\rho_{0}} - \frac{\tau_{hy}}{\rho_{0}} + hv_{s} \end{bmatrix} \end{aligned}$$

In Cartesian coordinates the system of 3D shallow water equations can be written

$$\frac{\partial U}{\partial t} + \frac{\partial F_x^{\ l}}{\partial x^{\prime}} + \frac{\partial F_y^{\ l}}{\partial y^{\prime}} + \frac{\partial F_{\sigma}^{\ l}}{\partial \sigma} + \frac{\partial F_x^{\ l}}{\partial x} + \frac{\partial F_y^{\ l}}{\partial y} + \frac{\partial F_{\sigma}^{\ l}}{\partial \sigma} = S$$
(3.12)

where the superscripts *I* and *V* denote the inviscid (convective) and viscous fluxes, respectively and where



$$\begin{aligned} \boldsymbol{U} &= \begin{bmatrix} h\\ hu\\ hv \end{bmatrix}, \\ \boldsymbol{F}_{x}^{I} &= \begin{bmatrix} h\overline{u} \\ hu^{2} + \frac{1}{2}g(h^{2} - d^{2}) \\ huv \end{bmatrix}, \quad \boldsymbol{F}_{x}^{V} &= \begin{bmatrix} 0 \\ hA\left(2\frac{\partial u}{\partial x}\right) \\ hA\left(\frac{\partial u}{\partial y} + \frac{\partial v}{\partial x}\right) \end{bmatrix} \\ \boldsymbol{F}_{y}^{I} &= \begin{bmatrix} h\overline{v} \\ hvu \\ hv^{2} + \frac{1}{2}g(h^{2} - d^{2}) \end{bmatrix}, \quad \boldsymbol{F}_{y}^{V} &= \begin{bmatrix} 0 \\ hA\left(\frac{\partial u}{\partial y} + \frac{\partial v}{\partial x}\right) \\ hA\left(2\frac{\partial v}{\partial y}\right) \end{bmatrix} \end{aligned}$$
(3.13)  
$$\boldsymbol{F}_{\sigma}^{I} &= \begin{bmatrix} h\omega \\ h\omega u \\ h\omega v \end{bmatrix}, \quad \boldsymbol{F}_{\sigma}^{V} &= \begin{bmatrix} 0 \\ \frac{v_{i}}{h}\frac{\partial u}{\partial \sigma} \\ \frac{v_{i}}{\partial \sigma} \\ \frac{v_{i}}{h}\frac{\partial v}{\partial \sigma} \end{bmatrix} \\ \boldsymbol{S} &= \begin{bmatrix} 0 \\ g\eta \frac{\partial d}{\partial x} + fvh - \frac{h}{\rho_{0}}\frac{\partial p_{a}}{\partial x'} - \frac{hg}{\rho_{0}}\int_{z}^{\eta}\frac{\partial \rho}{\partial y}dz - \frac{1}{\rho_{0}}\left(\frac{\partial s_{xx}}{\partial x} + \frac{\partial s_{xy}}{\partial y}\right) + hu_{s} \\ g\eta \frac{\partial d}{\partial y} - fuh - \frac{h}{\rho_{0}}\frac{\partial p_{a}}{\partial y'} - \frac{hg}{\rho_{0}}\int_{z}^{\eta}\frac{\partial \rho}{\partial y}dz - \frac{1}{\rho_{0}}\left(\frac{\partial s_{yx}}{\partial x} + \frac{\partial s_{yy}}{\partial y}\right) + hv_{s} \end{bmatrix}$$

Integrating Eq. (3.9) over the *i*th cell and using Gauss's theorem to rewrite the flux integral gives

$$\int_{A_i} \frac{\partial U}{\partial t} d\Omega + \int_{\Gamma_i} (F \cdot n) \, ds = \int_{A_i} S(U) d\Omega \tag{3.14}$$

where  $A_i$  is the area/volume of the cell  $\Omega$  is the integration variable defined on  $A_i$ ,  $\Gamma_i$  is the boundary of the *i*th cell and *ds* is the integration variable along the boundary. *n* is the unit outward normal vector along the boundary. Evaluating the area/volume integrals by a one-point quadrature rule, the quadrature point being the centroid of the cell, and evaluating the boundary intergral using a mid-point quadrature rule, Eq. (3.14) can be written



$$\frac{\partial U_i}{\partial t} + \frac{1}{A_i} \sum_{j}^{NS} \boldsymbol{F} \cdot \boldsymbol{n} \,\Delta \boldsymbol{\Gamma}_j = S_i \tag{3.15}$$

Here  $U_i$  and  $S_i$ , respectively, are average values of U and S over the *i*th cell and stored at the cell centre, NS is the number of sides of the cell,  $n_j$  is the unit outward normal vector at the *j*th side and  $\Delta\Gamma_j$  the length/area of the *j*th interface.

Both a first order and a second order scheme can be applied for the spatial discretization.

For the 2D case an approximate Riemann solver (Roe's scheme, see Roe, 1981) is used to calculate the convective fluxes at the interface of the cells. Using the Roe's scheme the dependent variables to the left and to the right of an interface have to be estimated. Second-order spatial accuracy is achieved by employing a linear gradient-reconstruction technique. The average gradients are estimated using the approach by Jawahar and Kamath, 2000. To avoid numerical oscillations a second order TVD slope limiter (Van Leer limiter, see Hirch, 1990 and Darwish, 2003) is used.

For the 3D case an approximate Riemann solver (Roe's scheme, see Roe, 1981) is used to calculate the convective fluxes at the vertical interface of the cells (x'y'-plane). Using the Roe's scheme the dependent variables to the left and to the right of an interface have to be estimated. Second-order spatial accuracy is achieved by employing a linear gradient-reconstruction technique. The average gradients are estimated using the approach by Jawahar and Kamath, 2000. To avoid numerical oscillations a second order TVD slope limiter (Van Leer limiter, see Hirch, 1990 and Darwish, 2003) is used. The convective fluxes at the horizontal interfaces (vertical line) are derived using first order upwinding for the low order scheme. For the higher order scheme the fluxes are approximated by the mean value of the fluxes calculated based on the cell values above and below the interface for the higher order scheme.

#### 3.1.3 Transport equations

The transport equations arise in the salt and temperature model, the turbulence model and the generic transport model. They all share the form of Equation Eq. (2.20) in Cartesian coordinates. For the 2D case the integral form of the transport equation can be given by Eq. (3.9) where

 $\boldsymbol{U} = h\overline{C}$ 

$$F^{\prime} = \begin{bmatrix} h\overline{u}\overline{C}, & h\overline{v}\overline{C} \end{bmatrix}$$
$$F^{\nu} = \begin{bmatrix} hD_{h}\frac{\partial\overline{C}}{\partial x}, & hD_{h}\frac{\partial\overline{C}}{\partial y} \end{bmatrix}$$

(3.16)

$$\mathbf{S} = -hk_{p}\overline{C} + hC_{s}S.$$

For the 3D case the integral form of the transport equation can be given by Eq. (3.9) where



 $\boldsymbol{U} = hC$ 

$$\boldsymbol{F}^{I} = \begin{bmatrix} huC, & hvC, & h\omegaC \end{bmatrix}$$
$$\boldsymbol{F}^{V} = \begin{bmatrix} hD_{h}\partial\frac{\partial C}{\partial x}, & hD_{h}\partial\frac{\partial C}{\partial y}, & h\frac{D_{h}}{h}\partial\frac{\partial C}{\partial \sigma} \end{bmatrix}$$
(3.17)

$$\mathbf{S} = -hk_pC + hC_sS.$$

The discrete finite volume form of the transport equation is given by Eq. (3.15). As for the shallow water equations both a first order and a second order scheme can be applied for the spatial discretization.

In 2D the low order approximation uses simple first order upwinding, i.e., element average values in the upwinding direction are used as values at the boundaries. The higher order version approximates gradients to obtain second order accurate values at the boundaries. Values in the upwinding direction are used. To provide stability and minimize oscillatory effects, a TVD-MUSCL limiter is applied (see Hirch, 1990, and Darwish, 2003).

In 3D the low order version uses simple first order upwinding. The higher order version approximates horizontal gradients to obtain second order accurate values at the horizontal boundaries. Values in the upwinding direction are used. To provide stability and minimize oscillatory effects, an ENO (Essentially Non-Oscillatory) type procedure is applied to limit the horizontal gradients. In the vertical direction a 3<sup>rd</sup> order ENO procedure is used to obtain the vertical face values (Shu, 1997).

## 3.2 Time Integration

Consider the general form of the equations

$$\frac{\partial \boldsymbol{U}}{\partial t} = \boldsymbol{G}(\boldsymbol{U}) \tag{3.18}$$

For 2D simulations, there are two methods of time integration for both the shallow water equations and the transport equations: A low order method and a higher order method. The low order method is a first order explicit Euler method

$$\boldsymbol{U}_{n+1} = \boldsymbol{U}_n + \Delta t \ \boldsymbol{G}(\boldsymbol{U}_n) \tag{3.19}$$

where  $\Delta t$  is the time step interval. The higher order method uses a second order Runge Kutta method on the form:

$$U_{n+\frac{1}{2}} = U_n + \frac{1}{2}\Delta t \ \boldsymbol{G}(U_n)$$

$$U_{n+1} = U_n + \Delta t \ \boldsymbol{G}(U_{n+\frac{1}{2}})$$
(3.20)



For 3D simulations the time integration is semi-implicit. The horizontal terms are treated implicitly and the vertical terms are treated implicitly or partly explicitly and partly implicitly. Consider the equations in the general semi-implicit form.

$$\frac{\partial U}{\partial t} = G_h(U) + G_v(BU) = G_h(U) + G_v^I(U) + G_v^V(U)$$
(3.21)

where the h and v subscripts refer to horizontal and vertical terms, respectively, and the superscripts refer to invicid and viscous terms, respectively. As for 2D simulations, there is a lower order and a higher order time integration method.

The low order method used for the 3D shallow water equations can written as

$$U_{n+1} - \frac{1}{2}\Delta t \left( G_{\nu}(U_{n+1}) + G_{\nu}(U_{n}) \right) = U_{n} + \Delta t \ G_{h}(U_{n})$$
(3.22)

The horizontal terms are integrated using a first order explicit Euler method and the vertical terms using a second order implicit trapezoidal rule. The higher order method can be written

$$U_{n+1/2} - \frac{1}{4} \Delta t \left( \boldsymbol{G}_{\nu}(\boldsymbol{U}_{n+1/2}) + \boldsymbol{G}_{\nu}(\boldsymbol{U}_{n}) \right) = \boldsymbol{U}_{n} + \frac{1}{2} \Delta t \ \boldsymbol{G}_{h}(\boldsymbol{U}_{n})$$

$$U_{n+1} - \frac{1}{2} \Delta t \left( \boldsymbol{G}_{\nu}(\boldsymbol{U}_{n+1}) + \boldsymbol{G}_{\nu}(\boldsymbol{U}_{n}) \right) = \boldsymbol{U}_{n} + \Delta t \ \boldsymbol{G}_{h}(\boldsymbol{U}_{n+1/2})$$
(3.23)

The horizontal terms are integrated using a second order Runge Kutta method and the vertical terms using a second order implicit trapezoidal rule.

The low order method used for the 3D transport equation can written as

$$\boldsymbol{U}_{n+1} - \frac{1}{2}\Delta t \left( \boldsymbol{G}_{\nu}^{\nu}(\boldsymbol{U}_{n+1}) + \boldsymbol{G}_{\nu}^{\nu}(\boldsymbol{U}_{n}) \right) = \boldsymbol{U}_{n} + \Delta t \boldsymbol{G}_{h}(\boldsymbol{U}_{n}) + \Delta t \boldsymbol{G}_{\nu}^{I}(\boldsymbol{U}_{n})$$
(3.24)

The horizontal terms and the vertical convective terms are integrated using a first order explicit Euler method and the vertical viscous terms are integrated using a second order implicit trapezoidal rule. The higher order method can be written

$$U_{n+1/2} - \frac{1}{4} \Delta t \left( G_{\nu}^{\nu}(U_{n+1/2}) + G_{\nu}^{\nu}(U_{n}) \right) = U_{n} + \frac{1}{2} \Delta t \ G_{h}(U_{n}) + \frac{1}{2} \Delta t \ G_{\nu}^{I}(U_{n})$$

$$U_{n+1} - \frac{1}{2} \Delta t \left( G_{\nu}^{\nu}(U_{n+1}) + G_{\nu}^{\nu}(U_{n}) \right) = U_{n} + \Delta t \ G_{h}(U_{n+1/2}) + \Delta t \ G_{\nu}^{I}(U_{n+1/2})$$
(3.25)

The horizontal terms and the vertical convective terms are integrated using a second order Runge Kutta method and the vertical terms are integrated using a second order implicit trapezoidal rule for the vertical terms.



## 3.3 Boundary Conditions

#### 3.3.1 Closed boundaries

Along closed boundaries (land boundaries), normal fluxes are forced to zero for all variables. For the momentum equations, this leads to full-slip along land boundaries. For the shallow water equations, the no slip condition can also be applied where both the normal and tangential velocity components are zero.

#### 3.3.2 Open boundaries

For the shallow water equations a number of different boundary conditions can be applied

The flux, velocity and Flather boundary conditions are all imposed using a weak approach. A ghost cell technique is applied where the primitive variables in the ghost cell are specified. The water level is evaluated based on the value of the adjacent interior cell, and the velocities are evaluated based on the boundary information. For a discharge boundary, the transverse velocity is set to zero for inflow and passively advected for outflow. The boundary flux is then calculated using an approximate Riemann solver.

The Flather (1976) condition is one of the most efficient open boundary conditions. It is very efficient in connection with downscaling coarse model simulations to local areas (see Oddo and Pinardi (2007)). The instabilities, which are often observed when imposing stratified density at a water level boundary, can be avoided using Flather conditions

The level boundary is imposed using a strong approach based on the characteristic theory (see e.g. Sleigh et al., 1998).

The discharge boundary condition is imposed using both a weak formulation using ghost cell technique described above and a strong approach based on the characteristic theory (see e.g. Sleigh et al., 1998).

Note that using the weak formulation for a discharge boundary the effective discharge over the boundary may deviate from the specified discharge.

For transport equations, either a specified value or a zero gradient can be given. For specified values, the boundary conditions are imposed by applying the specified concentrations for calculation of the boundary flux. For a zero gradient condition, the concentration at the boundary is assumed to be identical to the concentration at the adjacent interior cell.

#### 3.3.3 Flooding and drying

The approach for treatment of the moving boundaries problem (flooding and drying fronts) is based on the work by Zhao et al. (1994) and Sleigh et al. (1998). When the depths are small the problem is reformulated and only when the depths are very small the elements/cells are removed from the calculation. The reformulation is made by setting the momentum fluxes to zero and only taking the mass fluxes into consideration.

The depth in each element/cell is monitored and the elements are classified as dry, partially dry or wet. Also the element faces are monitored to identify flooded boundaries.



- An element face is defined as flooded if the following two criteria are satisfied: Firstly, the water depth at one side of face must be less than a tolerance depth,  $h_{dry}$ , and the water depth at the other side of the face larger than a tolerance depth,  $h_{flood}$ . Secondly, the sum of the still water depth at the side for which the water depth is less than  $h_{dry}$  and the surface elevation at the other side must be larger than zero.
- An element is dry if the water depth is less than a tolerance depth,  $h_{dry}$ , and no of the element faces are flooded boundaries. The element is removed from the calculation.
- An element is partially dry if the water depth is larger than  $h_{dry}$  and less than a tolerance depth,  $h_{wet}$ , or when the depth is less than the  $h_{dry}$  and one of the element faces is a flooded boundary. The momentum fluxes are set to zero and only the mass fluxes are calculated.
- An element is wet if the water depth is greater than  $h_{wet}$ . Both the mass fluxes and the momentum fluxes are calculated.

The wetting depth,  $h_{wet}$ , must be larger than the drying depth,  $h_{dry}$ , and flooding depth,  $h_{flood}$ , must satisfy

$$h_{dry} < h_{flood} < h_{wet} \tag{3.26}$$

The default values are  $h_{drv} = 0.005 \, m$ ,  $h_{flood} = 0.05 \, m$  and  $h_{wel} = 0.1 \, m$ .

Note, that for very small values of the tolerance depth,  $h_{wet}$ , unrealistically high flow velocities can occur in the simulation and give cause to stability problems.





## 4 Infiltration and Leakage

The effect of infiltration and leakage at the surface zone may be important in cases of flooding scenarios on otherwise dry land. It is possible to account for this in one of two ways: by Net infiltration rates or by constant infiltration with capacity.



Figure 4.1 Illustration of infiltration process

## 4.1 Net Infiltration Rates

The net infiltration rate is defined directly. This will act as a simple sink in each element in the overall domain area.

The one-dimensional vertical continuity equation is solved at each hydrodynamic time step after the two-dimensional horizontal flow equations have been solved. The calculation of the new water depth in the free surface zone for each horizontal element is found by

$$H(j) = H(j) - V_{infiltration}(j) / A(j)$$
(4.1)

Where  $V_{infiltration}(j)$  is the infiltrated volume in element (*j*) and A(j) the area of the element.

If H(j) becomes marked as *dry* then element (j) will be taken out of the two-dimensional horizontal flow calculations and no infiltration can occur until the element is flooded again.

In summary: when using Net infiltration rate an unsaturated zone is never specified and thus has no capacity limits, so the specified infiltration rates will always be fully effectuated as long as there is enough water available in the element.

## 4.2 Constant Infiltration with Capacity

Constant infiltration with capacity describes the infiltration from the free surface zone to the unsaturated zone and from the unsaturated zone to the saturated zone by a simplified model. The model assumes the following:

- The unsaturated zone is modelled as an infiltration zone with constant porosity over the full depth of the zone.
- The flow between the free surface zone and the infiltration zone is based on a constant flow rate, i.e.  $V_{infiltration} = Q_i \cdot \Delta t$  where  $Q_i$  is the prescribed flow rate.
- The flow between the saturated and unsaturated zone is modelled as a leakage  $Q_l$  having a constant flow rate, i.e.  $V_{leakage} = Q_l \cdot \Delta t$ .

The simplified model described above is solved through a one-dimensional continuity equation. Feedback from the infiltration and leakage to the two-dimensional horizontal hydrodynamic calculations is based solely on changes to the depth of the free surface zone – the water depth.

Note that the infiltration flow cannot exceed the amount of water available in the free surface water zone nor the difference between the water capacity of the infiltration zone and the actual amount of water stored there. It is possible that the infiltration flow completely drains the free surface zone from water and thus creates a dried-out point in the two-dimensional horizontal flow calculations.

The one-dimensional vertical continuity equation is solved at each hydrodynamic time step after the two-dimensional horizontal flow equations have been solved. The solution proceeds in the following way:

1. Calculation of the volume from leakage flow in each horizontal element  $-V_{leakage}(j)$ 

$$V_{leakage}(j) = Q_l(j) \cdot \Delta t \cdot A(j)$$
(4.2)

$$V_{leakage}(j) = \min(V_{leakage}(j), V_{i}(j))$$
(4.3)

$$V_{i}(j) := V_{i}(j) - V_{leakage}(j)$$

$$(4.4)$$

Where  $V_i(j)$  is the total amount of water in the infiltration zone and  $Q_l(j)$  is the leakage flow rate.

2. Calculation of the volume from infiltration flow in each horizontal element –  $V_{infiltration}(j)$ 

 $V_{infiltration}(j) = Q_i(j) \cdot \Delta t \cdot A(j)$ (4.5)

$$V_{infiltration}(j) = \min(V_{infiltration}(j), SC_i(j) - V_i(j), H(j) \cdot A(j))$$
(4.6)

$$V_{i}(j) := V_{i}(j) + V_{infiltration}(j)$$

$$(4.7)$$



Where  $Q_i(j)$  is the infiltration rate,  $SC_i(j)$  is the water storage capacity and H(j) the depth of the free surface.

 Calculation of the new water depth in the free surface zone for each horizontal element

$$H(j) = H(j) - V_{infiltration}(j)/A(j)$$
(4.8)

If H(j) becomes marked as *dry* then element (j) will be taken out of the two-dimensional horizontal flow calculations. The element can still *leak* but no infiltration can occur until the element is flooded again.

The water storage capacity of the infiltration zone is calculated as

$$SC_i(j) = Z_i(j) \cdot A(j) \cdot \gamma(j) \tag{4.9}$$

Where  $Z_i(j)$  is the depth of the infiltration zone and  $\gamma(j)$  is the porosity of the same zone.

In summary, when using Constant infiltration with capacity there can be situations where the picture is altered and the rates are either only partially effectuated or not at all:

- If =  $H(j) < H_{dry}$  on the surface (dry surface) => infiltration rate is not effectuated
- If: the water volume in the infiltration zone reaches the full capacity => infiltration rate is not effectuated
- If: the water volume is zero in the infiltration zone (the case in many initial conditions)
   => leakage rate is not effectuated
- Leakage volume must never eclipse the available water volume in the infiltration zone, if so we utilise the available water volume in infiltration zone as leakage volume
- Infiltration volume must never eclipse the available water volume on the surface, if so we utilise the available water on the surface as infiltration volume





## 5 Validation

The new finite-volume model has been successfully tested in a number of basic, idealised situations for which computed results can be compared with analytical solutions or information from the literature. The model has also been applied and tested in more natural geophysical conditions; ocean scale, inner shelves, estuaries, lakes and overland, which are more realistic and complicated than academic and laboratory tests. A detailed validation report is under preparation.

This chapter presents a comparison between numerical model results and laboratory measurements for a dam-break flow in an L-shaped channel.

Additional information on model validation and applications can be found here

http://www.mikepoweredbydhi.com/download/product-documentation

## 5.1 Dam-break Flow through Sharp Bend

The physical model to be studied combines a square-shaped upstream reservoir and an L-shaped channel. The flow will be essentially two-dimensional in the reservoir and at the angle between the two reaches of the L-shaped channel. However, there are numerical and experimental evidences that the flow will be mostly unidimensional in both rectilinear reaches. Two characteristics or the dam-break flow are of special interest, namely

- The "damping effect" of the corner
- The upstream-moving hydraulic jump which forms at the corner

The multiple reflections of the expansion wave in the reservoir will also offer an opportunity to test the 2D capabilities of the numerical models. As the flow in the reservoir will remain subcritical with relatively small-amplitude waves, computations could be checked for excessive numerical dissipation.

#### 5.1.1 Physical experiments

A comprehensive experimental study of a dam-break flow in a channel with a 90 bend has been reported by Frazão and Zech (2002, 1999a, 1999b). The channel is made of a 3.92 and a 2.92 metre long and 0.495 metre wide rectilinear reaches connected at right angle by a 0.495 x 0.495 m square element. The channel slope is equal to zero. A guillotine-type gate connects this L-shaped channel to a 2.44 x 2.39 m (nearly) square reservoir. The reservoir bottom level is 33 cm lower that the channel bed level. At the downstream boundary a chute is placed. See the enclosed figure for details.

Frazão and Zech performed measurements for both dry bed and wet bed condition. Here comparisons are made for the case where the water in the reservoir is initially at rest, with the free surface 20 cm above the channel bed level, i.e. the water depth in the reservoir is 53 cm. The channel bed is initially dry. The Manning coefficients evaluated through steady-state flow experimentation are 0.0095 and 0.0195 s/m<sup>1/3</sup>, respectively, for the bed and the walls of the channel.



The water level was measured at six gauging points. The locations of the gauges are shown in Figure 5.1 and the coordinates are listed in Table 5.1.



Figure 5.1 Set-up of the experiment by Frazão and Zech (2002)

Location	x (m)	y (m)
T1	1.19	1.20
T2	2.74	0.69
Т3	4.24	0.69
Τ4	5.74	0.69
Т5	6.56	1.51
Т6	6.56	3.01

Table 5.1	Location of the gauging poin	ts
1 4010 0.1	Ecoulien of the gauging point	

#### 5.1.2 Numerical experiments

Simulations are performed using both the two-dimensional and the three-dimensional shallow water equations.

An unstructured mesh is used containing 18311 triangular elements and 9537 nodes. The minimum edge length is 0.01906 m and the maximum edge length is 0.06125 m. In the 3D simulation 10 layers is used for the vertical discretization. The time step is 0.002 s. At the downstream boundary, a free outfall (absorbing) boundary condition is applied. The wetting depth, flooding depth and drying depth are 0.002 m, 0.001 m and 0.0001 m, respectively.

A constant Manning coefficient of 105.26 m<sup>1/3</sup>/s is applied in the 2D simulations, while a constant roughness height of 5.10<sup>-5</sup> m is applied in the 3D simulation.



#### 5.1.3 Results

In Figure 5.2 time series of calculated surface elevations at the six gauges locations are compared to the measurements. In Figure 5.3 contour plots of the surface elevations are shown at T = 1.6, 3.2 and 4.8 s (two-dimensional simulation).

In Figure 5.4 a vector plot and contour plots of the current speed at a vertical profile along the centre line (from (x,y)=(5.7, 0.69) to (x,y)=(6.4, 0.69)) at T = 6.4 s is shown.



Figure 5.2 Time evolution of the water level at the six gauge locations. (blue) 3D calculation, (black) 2D calculation and (red) Measurements by Frazão and Zech (1999a,b)





Figure 5.3 Contour plots of the surface elevation at T = 1.6 s (top), T = 3.2 s (middle) and T = 4.8 s (bottom).









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